

(19) 世界知的所有権機関  
国際事務局



(43) 国際公開日  
2004 年 1 月 22 日 (22.01.2004)

PCT

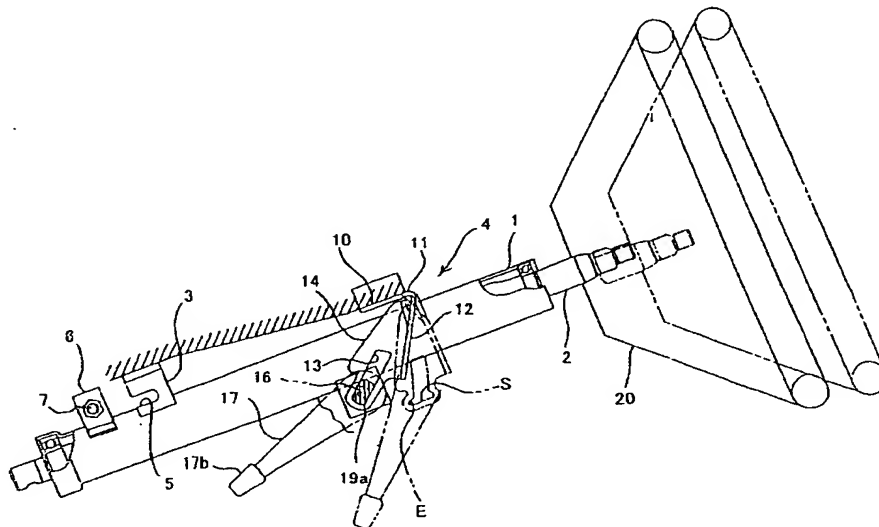
(10) 国際公開番号  
WO 2004/007261 A1

- (51) 国際特許分類<sup>7</sup>: B62D 1/19, B60R 21/05 (72) 発明者; および  
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特願2002-201511 2002 年 7 月 10 日 (10.07.2002) JP 都 中央区 日本橋 3 丁目 1 番 4 号 画廊ビル 3 階 Tokyo  
特願2002-271047 2002 年 9 月 18 日 (18.09.2002) JP (JP).  
特願2003-146697 2003 年 5 月 23 日 (23.05.2003) JP (81) 指定国 (国内): CN, US.  
特願2003-146710 2003 年 5 月 23 日 (23.05.2003) JP (84) 指定国 (広域): ヨーロッパ特許 (AT, BE, BG, CH, CY,  
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[続葉有]

(54) Title: VEHICLE SHOCK ABSORPTION TYPE STEERING COLUMN DEVICE

(54) 発明の名称: 車両用衝撃吸収式ステアリングコラム装置



(57) Abstract: In a vehicle shock absorption type steering column device capable of adjusting the steering position and adapted, at the time of a secondary collision, to absorb the shock energy while moving the steering column, which is supported by the vehicle through a bracket, forwardly of the vehicle, the bracket has a regulating section for regulating the steering position adjustment range of the steering column, the regulating section allowing, at the time of a secondary collision, the steering column to move beyond the steering adjustment range.

(57) 要約: ステアリング位置が調整可能であると共に、二次衝突時、ブラケットを介して車体に支持したステアリングコラムを車両前方に移動させつつ、その衝撃エネルギーを吸収する車両用衝撃吸収式ステアリング

[続葉有]



添付公開書類:

— 国際調査報告書

2文字コード及び他の略語については、定期発行される各PCTガゼットの巻頭に掲載されている「コードと略語のガイダンスノート」を参照。

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NSK2584PCTUS

## DESCRIPTION

IMPACT ABSORBING TYPE STEERING COLUMN APPARATUS FOR  
5 AUTOMOTIVE VEHICLE

Technical Field

The present invention relates to an impact  
absorbing type steering column apparatus for an  
10 automotive vehicle.

Background Arts

In case an automotive vehicle gets collided,  
there is a possibility in which a driver might  
15 encounter a secondary collision with a steering wheel  
due to inertia thereof. An impact absorbing type  
steering column apparatus is adopted in order to  
protect the driver on that occasion. A steering  
column is constructed so that an energy absorbing  
20 member, just when the driver secondarily collides  
with the steering wheel, gets collapsed while moving  
along a car body together with a steering shaft, and  
the steering column moves towards the front of the  
vehicle, thereby absorbing the impact energy thereof.

25 Given by way of one example of the impact  
absorbing type steering column apparatus is an impact  
energy absorbing system, wherein when the secondary

collision happens, as disclosed in Japanese Patent  
Publication No. 2978788 and Japanese Patent  
Application Laid-Open No. 2000-229577, the impact  
energy is absorbed by causing flexural deformations  
5 of car body sided brackets (a tilt bracket and a  
lower bracket) through which the steering column is  
attached to the car body.

By the way, as in the case of Japanese Patent  
Publication No. 2978788, upon the secondary collision,  
10 the tilt bracket gets deformed in flexure towards the  
front of the vehicle by dint of impact energy thereof.  
Then, a tilt fastening bolt gets displaced along a  
tilt adjusting groove and reaches a lowest position  
of this tilt adjusting groove, in which case this  
15 implies an arrival at a terminal of a collapse stroke  
of the energy absorbing member, and the steering  
column stops its collapsing movement.

Further, also as in the case of Japanese Patent  
Application Laid-Open No. 2000-229577, if the lower  
20 bracket makes a predetermined amount of its flexural  
deformation towards the front of the vehicle when the  
secondary collision occurs, it follows that the  
steering column reaches the terminal of its  
collapsing movement and stops its collapsing movement.

25 Thus, a normal operation of the steering column  
is that the steering column, generally when reaching  
the normally-set terminal of the collapsing movement,

stops and makes no further collapsing movement.

Moreover, there exists a demand for properly  
adjusting a moving range of the steering column  
against the secondary collision, corresponding to a  
5 type and a shipping destination of the automotive  
vehicle.

#### Disclosure of the Invention

It is an object of the present invention, which  
10 was devised under such circumstances, to provide an  
impact absorbing type steering column apparatus for  
an automotive vehicle that is capable of gaining a  
more extension of a moving range of a steering column  
against a secondary collision.

15 To accomplish the above object, according to a  
first invention, in an impact absorbing type steering  
column apparatus for an automotive vehicle, capable  
of adjusting a steering position and, when a  
secondary collision happens, absorbing impact energy  
20 thereof by moving a steering column supported through  
a bracket on a car body towards the front of the  
vehicle, there is provided an improvement  
characterized in that the bracket includes a  
restricting portion for restricting a steering  
25 position adjusting range of the steering column, and  
the restricting portion allows, upon the secondary  
collision, the steering column to move beyond the

steering position adjusting range.

According to the present invention, when the secondary collision happens, the steering column is capable of absorbing energy thereof while moving  
5 without being restricted within a range of the steering column position adjusting groove.

According to a second invention, in an impact absorbing type steering column apparatus for an automotive vehicle, capable of adjusting a steering  
10 position and, when a secondary collision happens, absorbing impact energy thereof by moving a steering column supported through a bracket on a car body towards the front of the vehicle, there is provided an improvement characterized in that the bracket  
15 includes a steering column position adjusting groove, through which a fastening member of the steering column is inserted and of which one end is opened, and a restricting portion for restricting a steering position adjusting range of the steering column, and  
20 the restricting portion allows, upon the secondary collision, the steering column to move beyond the steering position adjusting range.

According to the second invention, in the impact absorbing type steering column apparatus for the  
25 automotive vehicle, preferably, the groove serves for adjusting a tilt position of the steering column, and a lower bracket supporting the steering column

through a hinge mechanism in front of the vehicle and supported on the car body, is provided on a front-of-the vehicle side of the bracket, the lower bracket is provided with a cut-away portion through which a  
5 pivot of the hinge mechanism is inserted and of which a front-of-the-vehicle side is opened, and the pivot comes off the open side end of the cut-away portion by dint of a force acting in an axial direction of the steering column when the secondary collision  
10 happens, and the steering column is released from the lower bracket.

In the thus constructed impact absorbing type steering column apparatus, the pivot comes off the cut-away portion of the lower bracket by dint of the  
15 force acting in the axial direction, thereby enabling the steering column to get displaced without any restriction and enabling a collapse stroke to gain its further extension.

According to a third invention, in an impact  
20 absorbing type steering column apparatus for an automotive vehicle, capable of adjusting a steering position and, when a secondary collision happens, absorbing impact energy thereof by moving a steering column supported through a bracket on a car body  
25 towards the front of the vehicle, there is provided an improvement characterized in that there is provided a restricting member including a first

restricting portion and a second restricting portion,  
the restricting member allows, within the first  
restricting portion, the steering column to move for  
a positional adjustment, then deforms when the  
5 steering column moves, upon a secondary collision,  
beyond a first predetermined range restricted by the  
first restricting portion, and restricts the movement  
of the steering column within a second predetermined  
range by use of the second restricting portion.

10 Thus, according to the third invention, upon the  
secondary collision, the steering column, when moving  
up to the terminal of the first predetermined range  
as restricted by the first restricting portion, makes  
its further collapsing movement along the second  
15 restricting portion beyond the terminal in a way that  
causes the restricting member to deform. It is  
therefore possible to surely meet the demand for the  
proper adjustment, corresponding to the type and the  
shipping destination of the automotive vehicle.

20 According to a first aspect of the third  
invention, in the impact absorbing type steering  
column apparatus for the automotive vehicle, it is  
preferable that the bracket is constructed of an  
upper bracket and a lower bracket, a bolt is inserted  
25 through a hole of the upper bracket, and the steering  
column is supported by the upper bracket. It is also  
preferable that the restricting member is formed



integrally with the car body sided upper bracket, and the first restricting portion is formed with the hole. It is further preferable that when the steering column moves through only the first predetermined  
5 range upon the secondary collision, the bolt causes the restricting member to deform and enters the second restricting portion provided adjacent to the first restricting portion.

According to the first aspect of the third  
10 invention, in the impact absorbing type steering column apparatus for the automotive vehicle, preferably, when the bolt enters the second restricting portion, the restricting member makes its flexural deformation so as to extend in a moving  
15 direction of the bolt.

According to the first aspect of the third invention, in the impact absorbing type steering column apparatus for the automotive vehicle, preferably, the second restricting portion is  
20 previously formed as an elongate hole suitable for guiding the bolt in its moving direction when the bolt has entered the second restricting portion.

According to the first aspect of the third invention, in the impact absorbing type steering  
25 column apparatus for the automotive vehicle, preferably, the hole of the upper bracket is a groove for a tilt adjustment, and the bolt is a fastening

bolt for the tilt adjustment.

According to a second aspect of the third invention, in the impact absorbing type steering column apparatus for the automotive vehicle, it is preferable that the bracket is constructed of an upper bracket and a lower bracket, a bolt is inserted through a hole of the lower bracket, and the steering column is supported by the lower bracket. It is also preferable that the restricting member is formed integrally with the car body sided lower bracket, and the first restricting portion is formed with the hole. It is further preferable that when the secondary collision happens, impact energy is absorbed in a way that causes a flexural deformation of the restricting member while moving the steering column towards the front of the vehicle, and, when the steering column moves through only the first predetermined range, the bolt causes the restricting member to deform and enters the second restricting portion provided adjacent to the first restricting portion.

According to the second aspect of the third invention, in the impact absorbing type steering column apparatus for the automotive vehicle, it is preferable that when the bolt enters the second restricting portion, the restricting member makes its flexural deformation so as to extend in a moving direction of the bolt.

According to the second aspect of the third invention, in the impact absorbing type steering column apparatus for the automotive vehicle, it is preferable that the second restricting portion is  
5 previously formed as an elongate hole suitable for guiding the bolt in its moving direction when the bolt has entered the second restricting portion. According to the second aspect of the third invention, in the impact absorbing type steering column  
10 apparatus for the automotive vehicle, it is preferable that the hole of the car body sided lower bracket is a support hole for the tilt adjustment, and the bolt is a tilt adjusting hinge pin for determining a tilt center when inserted into the  
15 support hole.

#### Brief Description of the Drawings

FIG. 1 is a side view of an impact absorbing type steering column apparatus for an automotive  
20 vehicle according to a first embodiment of the present invention;

FIG. 2 is a plan view of the impact absorbing type steering column apparatus for the automotive vehicle shown in FIG. 1;

25 FIG. 3 is a sectional view taken along the line A-A in FIG. 1;

FIG. 4 is a sectional view taken along the line

B-B in FIG. 1;

FIG. 5 is an enlarged side view of a car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus for the automotive vehicle shown in FIG. 1;

FIG. 6 is a side view showing an early stage of a secondary collision in a state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in a real car;

FIG. 7 is a side view showing a later stage of the secondary collision in the state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in the real car;

FIG. 8 is a side view of a modified example of the impact absorbing type steering column apparatus for the automotive vehicle in the first embodiment of the present invention;

FIG. 9 is an enlarged side view of a car body sided upper bracket (tilt bracket) mounted in the impact absorbing type steering column apparatus for the automotive vehicle according to a second embodiment of the present invention;

FIG. 10 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a third embodiment of the present invention;

FIG. 11 is a side view of the impact absorbing

type steering column apparatus for the automotive vehicle according to a fourth embodiment of the present invention;

FIG. 12 is a side view of the car body sided lower bracket according to modified examples of the third and fourth embodiments;

FIG. 13 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a fifth embodiment of the present invention;

FIG. 14A is a sectional view taken along the line II-II in FIG. 13; FIG. 14B is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus for the automotive vehicle shown in FIG. 13;

FIG. 15 is a sectional view taken along the line III-III in FIG. 13;

FIG. 16 is a side view showing the early stage of the secondary collision in the state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in the real car;

FIG. 17 is a side view showing the later stage of the secondary collision in the state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in the real car;

FIG. 18 is an enlarged side view of the car body

sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus for the automotive vehicle according to a modified example of the fifth embodiment of the present invention;

5           FIG. 19 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a sixth embodiment of the present invention;

10           FIG. 20 is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus shown in FIG. 19;

15           FIG. 21A is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus according to a seventh embodiment of the present invention; FIG. 21B is a view showing a behavior of the car body sided upper bracket illustrated in FIG. 21A when secondary collision happens;

20           FIG. 22A is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus according to an eighth embodiment of the present invention; FIG. 22B is a view showing a behavior of the car body sided upper bracket illustrated in FIG. 22A when secondary collision happens;

25           FIG. 23 is a side view of the impact absorbing

type steering column apparatus for the automotive vehicle according to a ninth embodiment of the present invention; and

FIG. 24 is a sectional view taken along the line XII-XII in FIG. 23.

#### The Embodiments of the Invention

An impact absorbing type steering column apparatus for an automotive vehicle according to an embodiment of the present invention, will hereinafter be described with reference to the drawings.

##### (First Embodiment)

FIG. 1 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a first embodiment of the present invention. FIG. 2 is a plan view of the impact absorbing type steering column apparatus for the automotive vehicle shown in FIG. 1. FIG. 3 is a sectional view taken along the line A-A in FIG. 1. FIG. 4 is a sectional view taken along the line B-B in FIG. 1. FIG. 5 is an enlarged side view of a car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus for the automotive vehicle shown in FIG. 1. FIG. 6 is a side view showing an early stage of a secondary collision in a state where the impact absorbing type steering column apparatus for the automotive vehicle is

mounted in a real car. FIG. 7 is a side view showing a later stage of the secondary collision in the state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in  
5 the real car.

As shown in FIG. 1, a steering shaft 2, to which a steering wheel (FIG. 6) is attached at a right side end thereof in FIG. 1, is rotatably supported through bearings 1a, 1b within a steering column 1. The  
10 steering column 1 is secured to a strength member of the car body through a car body sided lower bracket 3 at its lower side end portion and through a car body sided upper bracket (tilt bracket) 4 taking substantially an L-shape as viewed from a side  
15 surface at its intermediate portion.

As illustrated in FIGS. 1 and 4, the car body sided lower bracket 3 includes a pair of car body fitting portions 3a, 3b horizontally extending. The lower bracket 3 also includes a pair of face-to-face  
20 plate portions 3c, 3d disposed right and left and extending substantially in vertical directions from the pair of car body fitting portions 3a, 3b.

A column sided lower bracket 6 of the steering column 1 is fixed by welding to a cylindrical outer  
25 peripheral surface of the steering column 1.

The column sided lower bracket 6 has face-to-face plate portions 6a, 6b that face the face-to-face



plate portions 3a, 3d of the car body sided lower bracket 6. The face-to-face plate portions 6a, 6b are formed with round holes 6c, 6d.

5 The face-to-face plate portions 6a, 6b of the column sided lower bracket 6 are so held (nipped) as to be slidable in between the face-to-face plate portions 3c, 3d of the car body sided lower bracket 3.

10 The face-to-face plate portions 3c, 3d of the car body sided lower bracket 3 are formed with cut-away portions 5a, 5b opened in the front of the vehicle.

15 A tilt center bolt 7 inserted through the round holes 6c, 6d of the column sided lower bracket 6 of the steering column 1, is secured into these cut-away portions 5a, 5b, and, with this contrivance, the steering column 1 becomes movable towards the front of the vehicle when the secondary collision occurs.

20 It should be noted that as a substitute configuration for the illustrative example given above, the car body sided lower bracket may be formed with the round holes, and the column sided lower bracket may be formed with the cut-away portions opened in the opposite direction, thereby configuring a release structure against the secondary collision.

25 As shown in FIGS. 1 and 3, the car body sided upper bracket 4 taking substantially the L-shape is constructed of a car body fitting portion 10

horizontally extending and fitted to the car body through a bolt, etc., a vertical wall portion 12 bent substantially in an L-shape from this car body fitting portion 10 through a bending portion 11, and  
5 a column fastening/fixing portion 14 erecting from the vertical wall portion 12 and having a tilt groove 13.

Note that the bending portion 11 extends towards the rear of the car body fitting portion 10, and the  
10 fixing portion 14 extends towards the front of the vertical wall portion 12 in this illustrative example.

A distance bracket (column sided upper bracket) 15 fixed by welding or the like to the steering column 1 is slidably provided between the column  
15 fastening/fixing portions 14, 14 of the car body sided upper bracket (tilt bracket) 4. A tilt position fastening bolt 16 is inserted through tilt adjusting grooves (which will hereinafter simply be called tilt grooves) 13 of the column  
20 fastening/fixing portions 14 and through round holes 15a, 15b formed in the distance bracket 15.

A nut 39 is screwed to a screw portion formed at an end portion of the tilt position fastening bolt 16, thereby attaining the fastening fixation.

25 As a fastening lever 17 attached to a proximal end portion of the fastening bolt 16 is swung, the car body sided upper bracket 4 is brought into press-

contact with and fixed to the distance bracket 15 or released from this press-contact, whereby the upper bracket 4 can be fastened or released therefrom.

Further, when a positional adjustment is completed, a  
5 grip portion 17b of the fastening lever 17 is disposed closer to the front side of the vehicle than the proximal end portion 17a thereof.

Moreover, as illustrated in FIG. 3, a cam lock mechanism is provided at the proximal end portion of  
10 the tilt position fastening bolt 16. This cam lock mechanism is provided with a first cam 18 rotating together with the fastening lever 17, and with a non-rotational second cam 19 engaging with and locking the first cam 18.

15 Further, the second cam 19 is formed with an elliptical protruded portion 19a (FIGS. 3 and 5) engaging with the tilt groove 13 to keep the second cam 19 non-rotational, and moving along the tilt groove 13 when making a tilt adjustment.

20 Note that the construction is adaptable to a screw-based fastening method other than the cam-based fastening method shown above.

Furthermore, as shown in FIGS. 2 and 3, the bending portion 11 of the car body sided upper  
25 bracket 4 is formed with reinforcing beads 11a. A flexural load applied when the secondary collision happens can be adjusted by changing sizes of these

reinforcing beads 11a.

As illustrated in FIG. 6, in a state where the impact absorbing type steering column apparatus for the automotive vehicle according to the present embodiment is mounted in the real car, the bending portion 11 and the vertical wall portion 12 are disposed closer to the vehicle rear side than the car body fitting portion 10 of the car body sided upper bracket 4, while the column fastening/fixing portion 14 is disposed closer to the vehicle front side than the vertical wall portion 12. With this arrangement, the tilt fastening bolt 16 is positioned downwardly of the bending portion 11 substantially in the vertical directions.

Moreover, according to this embodiment, as shown in FIG. 5, on a lower side of the tilt groove 13, there is formed a stopper S defined as a restricting member, on which the elliptical protruded portion 19a of the second cam 19 abuts and which is capable of deforming upon the secondary collision. It is to be noted that when making the tilt adjustment, the protruded portion 19a just abuts on the stopper S, however, the stopper S does not deform.

Formed between the stopper S and the tilt groove 13 is an extra stroke portion E taking an elongate hole extending in the front-and-rear directions of the vehicle, and serving as a second restricting

portion.

Note that two side portions 13a, 13b of the tilt groove 13 of the column fastening/fixing portion 14 are, as shown in FIG. 5, formed in concave shapes easy to cause the flexural deformation when the secondary collision occurs. Further, a width d of the extra stroke portion E is set substantially uniform throughout the concave portions 13a, 13b.

The stopper S, when in a normal tilt operation, functions as the stopper that receives the abutment of the elliptical protruded portion 19a of the second cam 19 and defines a tilt adjustment range. On the other hand, when the secondary collision happens, the stopper S defines a first predetermined range of a collapsing movement of the steering column 1, and operates to make its flexural deformation as a large load is applied thereto.

As shown in FIGS. 5 and 7, at a later stage of the secondary collision, the elliptical protruded portion 19a of the second cam 19 moves towards the front of the vehicle and comes into the press-contact with the stopper S with the result that the stopper S makes its flexural deformation. Then, the tilt fastening bolt 16 enters the extra stroke portion E as the second restricting portion and causes the flexural deformation of the extra stroke portion E so as to extend in a moving direction of the tilt

fastening bolt 16, and further moves over the first predetermined range, thereby absorbing impact energy thereof.

5       With the aforementioned construction taken, at an early stage of the secondary collision, as illustrated in FIG. 6, when a secondary collision load acts on the steering wheel 20 towards the front of the vehicle from the rear thereof, the steering column 1 tries to move towards the front of the vehicle together with the distance bracket 15 and the  
10       tilt fastening bolt 16.

      The tilt fastening bolt 16 moves, as shown in FIG. 6, down to the lowest position (which corresponds, however, to (a position of) the stopper  
15       S for defining the first predetermined range of the collapsing movement of the steering column in this embodiment) of the tilt groove 13.

      On the other hand, as shown in FIG. 6, the column sided lower bracket 6 and the tilt center bolt  
20       7 separate from the car body sided lower bracket 3 in such a way that the tilt center bolt 7 is removed out of the cut-away portions 5a, 5b of the car body sided lower bracket 3, and move towards the front of the vehicle.

25       On this occasion, an impact load on a driver acts substantially horizontally towards the front of the vehicle from the rear thereof. On the other hand,

the tilt fastening bolt 16 is disposed downwardly of the bending portion 11 substantially in the vertical directions, then starts moving with the bending portion 11 used as a fulcrum substantially in the horizontal direction, and subsequently rotates around the bending portion 11 (fulcrum).

With this contrivance, as shown in FIG. 6, in the impact absorbing type steering column apparatus for the automotive vehicle according to the present embodiment, the vertical wall portion 12 and the column fastening/fixing portion 14 of the car body sided upper bracket 4 get collapsed while making their flexural deformations so as to rotate around the bending portion 11 (fulcrum), thereby absorbing secondary impact energy.

Thus, according to the present embodiment, the bending portion 11 and the vertical wall portion 12 are disposed closer to the rear side of the vehicle than the car body fitting portion 10, while the column fastening/fixing portion 14 is disposed closer to the front side of the vehicle than the vertical wall portion 12. Owing to this layout, upon the secondary collision, the vertical wall portion 12 and the column fastening/fixing portion 14 of the car body sided upper bracket 4 start moving in the rotating direction around the bending portion 11 as the fulcrum, however, the moving direction is

substantially horizontal and substantially coincident with an input direction (which is substantially horizontal) of the impact load from the driver. Accordingly, the starting movement of the car body sided upper bracket 4 can be stabilized when the  
5 secondary collision happens.

Moreover, according to the present embodiment, the grip portion 17b of the fastening lever 17 is disposed closer to the front side of the vehicle than  
10 the proximal end portion 17a thereof. Besides, upon the secondary collision, as shown in FIG. 6, the fastening lever 17 follows the collapsing motion of the car body sided bracket 4 and moves towards the front of the vehicle while rotating. It is therefore  
15 possible to further enhance the safety of the fastening lever 17 against striking on knees of the driver.

At the later stage of the secondary collision, as shown in FIG. 7, the steering column 1 has a  
20 contrivance of being movable along the extra stroke portion E over the first predetermined range as follows.

At the end of the early stage of the secondary collision, the steering column 1 reaches a terminal  
25 of the first predetermined range of the collapsing movement. To be specific, as illustrated in FIG. 5, the tilt fastening bolt 16 moves down to the lowest



position of the tilt groove 13, and the elliptical protruded portion 19a of the second cam 19 is brought into the press-contact with the stopper S that has been defining the first predetermined range.

5           As indicated by a solid line in FIG. 7, at the later stage of the secondary collision, the elliptical protruded portion 19a comes into the press-contact with the stopper S, thereby causing the flexural deformation of the stopper S. Then, the  
10 tilt fastening bolt 16 enters the extra stroke portion E as the second restricting portion and causes the flexural deformation of the extra stroke portion E so as to extend in the moving direction of the tilt fastening bolt 16, and moves over the first  
15 predetermined range as being regulated by the extra stroke portion E defined as the second restricting portion and further into a second predetermined range, thereby absorbing impact energy thereof. Note that the early stage and the later stage of the secondary  
20 collision are within a series of continuous movements but do not imply different movements or operations.

          From what has been described so far, according to the present embodiment, the steering column 1, when reaching the terminal of the first predetermined  
25 range, causes the stopper S to deform and moves along the extra stroke portion E beyond the first predetermined range, whereby the impact energy

thereof can be absorbed.

Hence, depending on a type and a shipping destination of the automotive vehicle, it is possible to surely meet a demand that the stroke of the steering column 1 be extended over the stroke end thereof.

It should be noted that if a grease containing an extreme-pressure additive such as molybdenum disulfide, etc. is applied over the tilt groove 13, the elliptical protruded portion 19a can slide on the interior of the tilt groove 13 more effectively in the embodiment discussed above.

Further, the grease containing the extreme-pressure additive such as molybdenum disulfide, etc. may be applied also to between the distance bracket 15 and the column fastening/fixing portion 14 and between the column fastening/fixing portion 14 and the nut 39 or the second cam 19.

(Modified Example of First Embodiment)

FIG. 8 shows a modified example of the first embodiment discussed above. In a car body sided upper bracket (tilt bracket) 4' shown in FIG. 8, a bending portion 11' and a vertical wall portion 12' are integrally formed in the front of a car body fitting portion 10', and a column fastening/fixing portion 14' is disposed closer to the rear side of the vehicle than the vertical wall portion 12'.

Accordingly, in this modified example, the vertical wall portion 12' of the car body side upper bracket is closer to the front side of the vehicle than the vertical wall portion 12 in the first embodiment discussed above. Other portions have the same configurations as in the first embodiment illustrated in FIG. 1, and the same portions are shown by marking them with the same numerals and symbols, and their explanations are omitted.

According to this modified example, the vertical wall portion 12' is provided closer to the front side of the vehicle than the vertical wall portion 12 in the first embodiment, and hence, when the secondary collision happens, the swing range of the steering column can be enlarged, and the collapse stroke can be taken large.

(Second Embodiment)

FIG. 9 is an enlarged side view of the car body sided upper bracket (tilt bracket) attached to the impact absorbing type steering column apparatus for the automotive vehicle according to a second embodiment of the present invention.

In the second embodiment, as shown in FIG. 9, the stoppers S are formed as a pair of protrusions (lumps) facing each other. The extra stroke portion E as the second restricting portion is previously formed as an elongate hole suitable for guiding the

tilt fastening bolt 16 in its moving direction when the secondary collision occurs. Other configurations and operations are the same as in the first embodiment discussed above.

5           Note that an interval between front side ends of the pair of protruded stoppers S is set narrower than a width of the elliptical protruded portion 19a.

          From the above, according to the second embodiment, at the end of the early stage of the  
10       secondary collision, the steering column 1 is restricted by the first restricting portion of the restricting member, and thus reaches the terminal of the first predetermined range of the collapsing movement. Namely, as shown in FIG. 9, the tilt  
15       fastening bolt 16 moves down to the lowest position of the tilt groove 13, and the elliptical protruded portion 19a of the second cam 19 is brought into the press-contact with the stoppers S serving as the restricting members.

20           At the later stage of the secondary collision, the elliptical protruded portion 19a comes into the press-contact with the stoppers S and intrudes itself over the stoppers S (or causes the flexural  
          deformations of the stoppers S). Hereupon, the tilt  
25       fastening bolt 16 enters the extra stroke portion E defined as the second restricting portion and moves along this extra stroke portion E.

Note that the extra stroke portion E may take its elongate hole shape configured by combining the elongate hole extending in the front-and-rear direction as adopted in the first embodiment with the  
5 elongate hole extending substantially in the vertical directions as adopted in the second embodiment. In this case, the collapsing movement can gain a further extension.

(Third Embodiment)

10 FIG. 10 is a side view of the impact absorbing type steering column apparatus according to a third embodiment of the present invention.

In the third embodiment, as illustrated in FIG. 10, a flange of the car body fitting portion 10 of  
15 the car body sided upper bracket (tilt bracket) 4 is provided with a release capsule 21 made of a resin, for releasing the car body sided upper bracket 4 from the car body upon the secondary collision. With this contrivance, upon the secondary collision, the  
20 operation is not that the car body sided upper bracket 4 makes the flexural deformation as seen in the embodiments discussed above but that the upper bracket 4 releases from the car body and moves forwards.

25 A car body sided lower bracket 30 is of such a type as to absorb the impact energy by making the flexural deformation. The car body sided lower

bracket 30 is constructed of a car body fitting  
portion 31 fitted to the car body by use of bolts,  
etc., a vertical wall portion 33 extending from the  
car body fitting portion 31, bent substantially in an  
5 L-shape through a bending portion 32 and disposed in  
the rear of the car body fitting portion 31, a bolt  
fixing plate portion 37 formed in front of the  
vertical wall portion 33, and so on.

Further, the grease containing the extreme-  
10 pressure additive such as molybdenum disulfide, etc.  
may be applied to between the stopper S, the hole of  
the extra stroke portion E, the bolt fastening/fixing  
portion 14 and a column sided lower bracket 36.

The car body sided lower bracket 30 is formed  
15 with a support hole 34 for a tilt hinge. A tilt  
hinge pin 35 for determining a tilt center is  
inserted in between the support hole 34 and the  
column sided lower bracket 36 of the steering column  
1.

20 Moreover, according to the third embodiment, the  
stopper S is formed on the lower side of the support  
hole 34. The stopper S, on which the tilt hinge pin  
35 abuts, deforms when the secondary collision  
happens.

25 The extra stroke portion E formed with an  
elongate hole extending in the front-and-rear  
directions of the vehicle, is formed between the

stopper S and the support hole 34.

The stopper S as the restricting member, when in the normal tilt operation, functions as the stopper that abuts on the tilt hinge pin 35 and holds it  
5 within the support hole 34. On the other hand, when the secondary collision happens, the stopper S defines the first predetermined range of the collapsing movement of the steering column 1, and operates to make its flexural deformation as a large  
10 load is applied thereto.

More specifically, at the later stage of the secondary collision, the tilt hinge pin 35, after causing the deformation of the stopper S, enters the extra stroke portion E and causes this extra stroke  
15 portion E to deform so as to extend in a moving direction of the tilt hinge pin 35, thereby absorbing the impact energy.

With the thus-contrived construction, at the early stage of the secondary collision, when the  
20 secondary impact load acts on the steering wheel 20 towards the front of the vehicle from the rear thereof, the steering column 1 tries to move towards the front of the vehicle together with the car body sided upper bracket 4 and the distance bracket 15 by  
25 dint of an operation of the release capsule 21.

On this occasion, the impact load on the driver acts substantially horizontally towards the front of

the vehicle from the rear thereof. On the other hand,  
the tilt hinge pin 35 is disposed downwardly of the  
bending portion 32 substantially in the vertical  
directions, then starts moving with the bending  
5 portion 32 used as a fulcrum substantially in the  
horizontal direction, and subsequently rotates around  
the bending portion 32 (fulcrum).

With this contrivance, the vertical wall portion  
33 gets collapsed while making the flexural  
10 deformation so as to rotate around the bending  
portion 32 (fulcrum), thereby absorbing the secondary  
impact energy.

Next, at the later stage of the secondary  
collision, the steering column 1 moves over the first  
15 predetermined range defined by the first restricting  
portion and can further move along the extra stroke  
portion E as the second restricting portion for  
defining a second predetermined range.

At the end of the early stage of the secondary  
20 collision, the steering column 1 reaches the terminal  
of the first predetermined range. Namely, the tilt  
hinge pin 35 is brought into the press-contact with  
the stopper S.

At the later stage of the secondary collision,  
25 the tilt hinge pin 35 comes into the press-contact  
with the stopper S with the result that the stopper S  
is made to deform. Then, the tilt hinge pin 35



enters the extra stroke portion E and causes the flexural deformation of the extra stroke portion E so as to extend in the moving direction of the tilt hinge pin 35, and, this being done, moves over the terminal portion of the first predetermined range, thereby absorbing the impact energy thereof. Note that the early stage and the later stage of the secondary collision are within a series of continuous movements but do not imply different movements or operations.

From what has been described so far, according to the third embodiment, the steering column 1, when reaching the terminal of the first predetermined range equal to the normally-set collapsing movement, causes the deformation of the stopper S as the restricting member that has formed this terminal and moves along the extra stroke portion E as the second restricting portion beyond the this terminal, whereby the impact energy thereof can be absorbed.

Hence, depending on a type and a shipping destination of the automotive vehicle, it is possible to surely meet a demand that the movement (stroke) of the steering column 1 be extended over the stroke end thereof.

It should be noted that the third embodiment may be combined with the first embodiment or the second embodiment.

(Fourth Embodiment)

FIG. 11 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a fourth embodiment of the present invention.

In the fourth embodiment, as shown in FIG. 11, the stoppers S as the restricting members are formed as a pair of protrusions (lumps) facing each other, and the extra stroke portion E configuring the second restricting portion is previously formed as an elongate hole extending obliquely downwards so as to guide the tilt hinge pin 35 in its moving direction when the secondary collision occurs. Other configurations and operations are the same as in the embodiments discussed above.

From the above, according to the fourth embodiment, at the end of the early stage of the secondary collision, the steering column 1 reaches the terminal of the first predetermined range while making the collapsing movement as restricted by one restricting portion. To be specific, the tilt hinge pin 35 is brought into the press-contact with the stoppers S configuring the restricting members.

At the later stage of the secondary collision, the tilt hinge pin 35 comes into the press-contact with the stoppers S and causes the stoppers S to deform. Then, the tilt hinge pin 35 enters the extra

stroke portion E configuring the second restricting portion of the restricting member, and can further move along the extra stroke portion E.

Note that the extra stroke portion E may take  
5 its elongate hole shape configured by combining the elongate hole extending in the front-and-rear direction as adopted in the third embodiment with the elongate hole extending substantially in the vertical directions as adopted in the present fourth  
10 embodiment. In this case, the collapsing movement can gain a further extension.

Moreover, the fourth embodiment may be combined with the first embodiment or the second embodiment.

It is to be noted that the present invention is  
15 not limited to the embodiments discussed above and can be modified in a variety of forms.

For example, in the first through fourth embodiments discussed above, the vertical wall portion may be disposed in front or in rear of the  
20 car body fitting portion without being limited to the upper and lower side brackets, and the column fastening/fixing portion may be disposed in front or in rear of the vertical wall portion.

Further, in the embodiments discussed above, the  
25 extra stroke may be provided on one side or both sides of the column fastening/fixing portion. In the case of providing the extra stroke on only one side

thereof, the tilt groove on the other side of the column fastening/fixing portion may be opened on its lower side or may be extended without providing the energy absorbing portion, and so on.

5           Moreover, as shown in FIG. 12, in a steering column apparatus capable of making a telescopic adjustment, a bolt fixing plate portion 37 of the car body sided lower bracket 30 employed in the third and fourth embodiments, may be formed with an elongate  
10           hole 38 for the telescopic adjustment, a pair of protruded stoppers S and the extra stroke portion E (the second restricting portion).

          As discussed above, according to the fourth embodiment, upon the secondary collision, the  
15           steering column, when moving up to the terminal of the first predetermined range as being restricted by the first restricting portion, causes the restricting member to deform and further makes its collapsing movement along the second restricting portion beyond  
20           this terminal. Hence, depending on a type and a shipping destination of the automotive vehicle, it is possible to surely meet a demand for the proper adjustment.

(Fifth Embodiment)

25           FIG. 13 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a fifth embodiment of the

present invention.

FIG. 14A is a sectional view taken along the line II-II in FIG. 13. FIG. 14B is an enlarged side view of a car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus shown in FIG. 13.

FIG. 15 is a sectional view taken along the line III-III in FIG. 13.

FIG. 16 is a side view showing an early stage when a secondary collision happens in a state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in a real car.

FIG. 17 is a side view showing a later stage when the secondary collision happens in the state where the impact absorbing type steering column apparatus for the automotive vehicle is mounted in the real car.

As shown in FIG. 13, the steering shaft 2, to which a steering wheel 20 (FIG. 16) is attached at a right side end thereof, is rotatably supported through a pair of bearings 1a, 1b within the steering column 1. The steering column 1 is secured to a strength member of the car body through the car body sided lower bracket 3 at its lower side end portion and through the car body sided upper bracket (tilt bracket) 4 taking substantially an L-shape as viewed from a side surface at its intermediate portion.

As illustrated in FIGS. 13 and 16, the car body sided lower bracket 3 includes the pair of car body fitting portions 3a, 3b. The lower bracket 3 also includes the pair of face-to-face plate portions 3c, 3d disposed right and left and extending substantially in a vertical direction from the pair of car body fitting portions 3a, 3b.

The column sided lower bracket 6 of the steering column 1 is fixed by welding to a cylindrical outer peripheral surface of the steering column 1.

The column sided lower bracket 6 has the face-to-face plate portions 6a, 6b that face the face-to-face plate portions 3a, 3d of the car body sided lower bracket 6. The face-to-face plate portions 6a, 6b are formed with the round holes 6c, 6d.

The face-to-face plate portions 6a, 6b of the column sided lower bracket 6 are so held (nipped) as to be slidable in between the face-to-face plate portions 3c, 3d of the car body sided lower bracket 3.

The face-to-face plate portions 3c, 3d of the car body sided lower bracket 3 are formed with the cut-away portions 5a, 5b opened in the front of the vehicle.

The tilt center bolt 7 inserted through the round holes 6c, 6d of the column sided lower bracket 6 of the steering column 1, is secured into these cut-away portions 5a, 5b, and, with this contrivance,

the steering column 1 becomes movable towards the front of the vehicle when the secondary collision occurs.

It should be noted that as a substitute  
5 configuration for the illustrative example given in the fifth embodiment, the car body sided lower bracket may be formed with the round holes, and the column sided lower bracket may be formed with the cut-away portions opened in the opposite direction,  
10 thereby configuring a release structure against the secondary collision.

As shown in FIGS. 13 and 15, the car body sided upper bracket 4 taking substantially the L-shape is constructed of the car body fitting portions 10, 10  
15 fitted to the strength member of the car body through bolts, etc., the vertical wall portions 12, 12 bent substantially in the L-shape from the car body fitting portions 10, 10 through the bending portions 11, 11, and the column fastening/fixing portions 14,  
20 14 erecting from the vertical wall portions 12, 12 and having the tilt grooves 13, 13 for the tilt adjustment.

Note that the bending portions 11, 11 are disposed in rear of the car body fitting portions 10, 10, while the column fastening/fixing portions 14, 14  
25 are disposed in front of the vertical wall portions 12, 12 in the present illustrative example in the

fifth embodiment.

The distance bracket (column sided upper bracket) 15 fixed by welding or the like to the steering column 1 is slidably provided between the column fastening/fixing portions 14, 14 of the car  
5 body sided upper bracket (tilt bracket) 4. The fastening bolt 16 is inserted through the tilt adjusting grooves 13 of the column fastening/fixing portions 14 and through the round holes 15a, 15b  
10 formed in the distance bracket 15.

The nut 39 is screwed to the screw portion formed at the side end portion of this fastening bolt 16, thereby attaining the fastening fixation.

As the fastening lever 17 attached to the proximal end portion of the fastening bolt 16 is  
15 swung, the car body sided upper bracket 4 is brought into the press-contact with and fixed to the distance bracket 15 or released from this press-contact, whereby the upper bracket 4 can be fastened or  
20 released therefrom. Further, when the positional adjustment is completed, the grip portion 17b of the fastening lever 17 is disposed closer to the front side of the vehicle than the proximal end portion 17a thereof.

25 Moreover, as illustrated in FIG. 14A, a cam lock mechanism is provided at the proximal end portion of the fastening bolt 16. This cam lock mechanism is



provided with the first cam 18 rotating together with the fastening lever 17, and with the non-rotational second cam 19 engaging with and locking the first cam 18.

5           Further, the second cam 19 is formed with the elliptical protruded portion 19a (FIG. 14A) engaging with the tilt adjusting groove 13 to keep the second cam 19 non-rotational, and moving along the tilt adjusting groove 13 when making a tilt adjustment.

10           Note that the construction is adaptable to a screw-based fastening method other than the cam-based fastening method shown above.

          Furthermore, as shown in FIGS. 14A and 14B, the bending portion 11 of the car body sided upper  
15    bracket 4 is formed with the reinforcing beads 11a. A flexural load applied when the secondary collision happens can be adjusted by changing sizes of these reinforcing beads 11a.

          As illustrated in FIG. 16, in a state where the  
20    impact absorbing type steering column apparatus for the automotive vehicle according to the fifth embodiment is mounted in the real car, the steering column apparatus is attached generally with a tilt to the car body. In this case, however, a tilt angle  
25    preferable to the operability by the driver of an automobile is within a range of 20° through 30°. In the real car, the bending portion 11 and the vertical

wall portion 12 are disposed closer to the rear side of the vehicle than the car body fitting portion of the car body sided upper bracket 4, while the column fastening/fixing portion 14 is disposed closer to the front side of the vehicle than this vertical wall portion 12. With this arrangement, the tilt fastening bolt 16 is positioned downwardly of the bending portion 11 substantially in the vertical direction.

10           In the present fifth embodiment, the tilt adjusting groove 13 is formed substantially in a circular arc with some tilt, and one side end thereof is an open side end 13a.

15           A protrusion (stopper) S protruding forwardly of car body so as to delimit a tilt adjusting range of the fastening bolt 16, is provided on the rear-of-the-vehicle side of this tilt adjusting groove 13.

20           This protrusion S has an abutting surface Sa on the side that faces the fastening bolt 16, and this abutting surface Sa functions as a stopper for delimiting a lower side end position of the tilt adjusting range of the fastening bolt 16.

25           Further, the protrusion (stopper) S is, upon the secondary collision, bent or broken, etc. by the fastening bolt 16, thereby allowing the fastening bolt 16 to move towards the front of the vehicle. With this contrivance, at the later stage of the

secondary collision, the fastening bolt 16 comes off the open side end 13a of the tilt adjusting groove 13, thereby enabling the steering column 1 to be released from the car body. Accordingly, before the bending  
5 portion 11 of the car body sided upper bracket 4 reaches a limit of the flexure, the steering column 1 can be released from the car body, and a sufficient collapse stroke can be ensured.

Note that a width of the open end 13a of the  
10 tilt adjusting groove 13 under the protrusion (stopper) S is, as is well illustrated in FIG. 14B, larger than its width above the protrusion S. Namely, a recessed portion d is formed under the protrusion S. With this configuration, the protrusion (stopper) S  
15 can escape into the recessed portion d after being bent downwards, i.e., counterclockwise in the illustrative example.

Further, the absorbable impact load can be controlled by changing a dimension of the width of  
20 the protrusion (stopper) S substantially in the vertical direction.

Still further, the protrusion (stopper) S is provided on the rear-of-the-vehicle side of the tilt adjusting groove 13, and may also be provided on the  
25 front-of-the-vehicle side of this groove 13. The protrusions (stoppers) S may also be provided on both sides of the tilt adjusting groove 13.

Yet further, the protrusion (stopper) S may also be provided in a cut-away portion 5a in the car body sided lower bracket 3.

Next, an operation in the fifth embodiment will  
5 be explained.

When fastened in the tilt adjusting position, upon rotating the fastening lever 17, the first cam 18 and the second cam 19 make relative displacements, and the fastening bolt 16 is fastened in an axial  
10 direction. Then, an interval between the pair of face-to-face plate portions 11, 11 of the car body sided bracket 4 is narrowed down, and these plate portions 11, 11 are brought into the press-contact with both of wall surfaces of the distance bracket 15.  
15 The steering column 1 is thus fastened.

By contrast, when rotating the fastening lever 17 in the reversed direction, the first cam 18 and the second cam 19 make the relative displacements, thereby releasing the fastening bolt 16 from being  
20 fastened in the axial direction. The pair of face-to-face plate portions 11, 11 is thereby released from the press-contact with both of the wall surfaces of the distance bracket 15, with the result that the steering column 1 is released from being fastened and  
25 can be tilted at a desired tilt angle.

Next, upon the secondary collision, at the early stage thereof, as shown in FIG. 16, when the

secondary impact load acts on the steering wheel 20 towards the front of the vehicle from the rear thereof, the steering column 1 tends to move towards the front of the vehicle together with the distance bracket 15 and the tilt fastening bolt 16.

Note that the tilt fastening bolt 16 moves, as illustrated in FIG. 16, down to the lowest position of the tilt adjusting groove 13, i.e., moves till it abuts on the protrusion (stopper) S.

Further, as shown in FIG. 16, the column sided lower bracket 6 and the tilt center bolt 7 separate from the car body sided lower bracket 3 in such a way that the tilt center bolt 7 is removed out of the cut-away portions 5a, 5b of the car body sided lower bracket 3, and move towards the front of the vehicle.

On this occasion, an impact load on the driver acts substantially horizontally towards the front of the vehicle from the rear thereof. On the other hand, the tilt fastening bolt 16 is disposed downwardly of the bending portion 11 substantially in the vertical direction, then starts moving in the horizontal direction with the bending portion 11 being made as a fulcrum substantially, and subsequently rotates around the bending portion 11 (fulcrum).

With this contrivance, as shown in FIG. 16, in the impact absorbing type steering column apparatus for the automotive vehicle according to the fifth

embodiment, the vertical wall portion 12 and the column fastening/fixing portion 14 of the car body sided upper bracket 4 get collapsed while making their flexural deformations so as to rotate around the bending portion 11 (fulcrum), thereby absorbing  
5 the secondary impact energy.

Thus, according to the fifth embodiment, the bending portion 11 and the vertical wall portion 12 are disposed closer to the rear side of the vehicle  
10 than the car body fitting portion 10, while the column fastening/fixing portion 14 is disposed closer to the front side of the vehicle than the vertical wall portion 12. Owing to this layout, upon the secondary collision, the vertical wall portion 12 and  
15 the column fastening/fixing portion 14 of the car body sided upper bracket 4 start moving in the rotating direction around the bending portion 11 as the fulcrum, however, the moving direction is substantially horizontal and substantially coincident  
20 with an input direction (which is substantially horizontal) of the impact load from the driver. Accordingly, the starting movement of the car body sided upper bracket 4 can be stabilized when the secondary collision happens.

25 Moreover, according to the fifth embodiment, the grip portion 17b of the fastening lever 17 is disposed closer to the front side of the vehicle than

the proximal end portion 17a thereof. Besides, upon the secondary collision, as shown in FIG. 4, the fastening lever 17 follows the collapsing motion of the car body sided bracket 4 and moves towards the front of the vehicle while rotating. It is therefore possible to further enhance the safety of the fastening lever 17 against striking on knees of the driver.

Next, at the later stage of the secondary collision, as shown in FIG. 17, the car body sided upper bracket 4 consecutively gets collapsed while making its flexural deformation so as to rotate around the bending portion 11 (fulcrum), thereby absorbing the secondary impact energy.

Simultaneously, the steering column 1 is so constructed as to be further movable beyond the tilt adjusting range as follows.

Namely, at the later stage of the secondary collision, the protrusion (stopper) S is bent or broken, etc. by the fastening bolt 16, thereby allowing the fastening bolt 16 to move towards the front of the vehicle. With this contrivance, at the later stage of the secondary collision, the fastening bolt 16 comes off the open end 13a of the tilt adjusting groove 13, thereby enabling the steering column 1 to be released from the car body. Accordingly, before the bending portion 11 of the car

body sided upper bracket 4 reaches a limit of the flexure, the steering column 1 can be released from the car body, and a sufficient collapse stroke can be ensured.

5           Note that the early stage and the later stage of the secondary collision are within a series of continuous movements but do not imply separate operations. Further, in the fifth embodiment discussed above, the grease containing the extreme-  
10 pressure additive such as molybdenum disulfide, etc. may be applied to the tilt adjusting groove 13 and the bracket slide surface, whereby the fastening bolt 16 can slide on inside the tilt adjusting groove 13 more effectively.

15           Moreover, the grease containing the extreme-pressure additive such as molybdenum disulfide, etc. may be applied to between the column fastening/fixing portion 14 of the car body sided upper bracket 4 and the nut 39 or the second cam 19, and between the bolt  
20 fastening/fixing portion 14 and the distance bracket 15.

(Modified Example of Fifth Embodiment)

FIG. 18 is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact  
25 absorbing type steering column apparatus according to a modified example of the fifth embodiment.

The protrusion (stopper) S provided in the tilt



adjusting groove 13 may consist of plural pieces of protrusions. As shown in FIG. 18, for instance, three pieces of protrusions (stoppers) S1, S2, S3 are formed in the tilt adjusting groove 13 uniformly in alignment in their directions towards the front the car body. Note that the upper protrusion S1 has an abutting surface Sa on the side facing the fastening bolt 16, and this abutting surface Sa functions as a stopper for delimiting a lower side end position of the tilt adjusting range of the fastening bolt 16.

In this modified example, at the later stage of the secondary collision, the fastening bolt 16 collapses sequentially the three protrusions (stoppers) S1, S2, S3 and thereafter comes off the open end 13a of the tilt adjusting groove 13. In this process, the energy can be absorbed through not only the deformation of the car body sided upper bracket 4 itself but also the deformations of the protrusions (stoppers) S1, S2, S3.

It is to be noted that recessed portions d1, d2 are formed respectively under the protrusions (stoppers) S1, S2. With this configuration, the protrusions (stoppers) S1, S2 can escape into the recessed portions d1, d2 after being bent downwards, i.e., counterclockwise in the illustrative example.

Further, the absorbable impact load can be controlled by changing a dimension of the width of

each of the protrusions (stoppers) S1, S2, S3 substantially in the vertical direction.

Still further, the protrusions (stoppers) S1, S2, S3 are provided on the rear-of-the-vehicle side of the tilt adjusting groove 13, and may also be provided on the front-of-the-vehicle side of this groove 13. The protrusions (stoppers) S1, S2, S3 may also be provided on both sides of the tilt adjusting groove 13.

10 (Sixth Embodiment)

FIG. 19 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a sixth embodiment of the present invention.

15 FIG. 20 is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus shown in FIG. 19.

In the sixth embodiment, a bending portion 11' and vertical wall portion 12' are integrally formed in the front of a car body fitting portion 10' in a car body sided upper bracket (tilt bracket) 4'. Column fastening/fixing portion 14' is disposed closer to the rear of the vehicle than the vertical wall portion 12'. Furthermore, the bending portion 11' of the car body sided upper bracket 4' is formed with a reinforcing bead 11a'. A flexural load

applied when the secondary collision happens can be adjusted by changing a size of the reinforcing bead 11a'.

Accordingly, in the sixth embodiment, the  
5 vertical wall portion 12' of the car body sided upper bracket 4' is disposed closer to the front side of the vehicle than the vertical wall portion 12 in the fifth embodiment described above. Other portions have the same configurations as in the fifth  
10 embodiment illustrated in FIG. 13, and the same portions are shown by marking them with the same numerals and symbols, and their explanations are omitted.

According to the sixth embodiment, the vertical  
15 wall portion 12' is provided closer to the front side of the vehicle than the vertical wall portion 12 in the fifth embodiment, and hence, when the secondary collision happens, the swing range of the steering column 1 can be enlarged, and the collapse stroke can  
20 be taken large.

Note that a width of the open end 13a of the tilt adjusting groove 13 under the protrusion (stopper) S is, as is well illustrated in FIG. 20, larger than its width above the protrusion S. Namely,  
25 a recessed portion d is formed under the protrusion S. With this configuration, the protrusion (stopper) S can escape into the recessed portion d after being

bent downwards, i.e., counterclockwise in the illustrative example.

Further, the absorbable impact load can be controlled by changing a dimension of the width of the protrusion (stopper) S substantially in the vertical direction.

Still further, the protrusion (stopper) S is provided on the rear-of-the-vehicle side of the tilt adjusting groove 13, and may also be provided on the front-of-the-vehicle side of this groove 13. The protrusions (stoppers) S may also be provided on both sides of the tilt adjusting groove 13.

(Seventh Embodiment)

FIG. 21A is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus according to a seventh embodiment of the present invention. FIG. 21B is a view showing a behavior of the car body sided upper bracket illustrated in FIG. 21A when secondary collision happens.

In the seventh embodiment, the car sided upper bracket 4 is formed with a restricting member 30 extending substantially towards the front of the vehicle so as to form a gap g between the open end 13a of the tilt adjusting groove 13 and the restricting member 30 itself. This restricting member 30 delimits substantially the lower end of the

tilt adjusting range of the tilt adjusting groove 13.

Further, the gap  $g$  is set smaller than a diameter of the fastening bolt 16. Moreover, the restricting member 30 is formed with a bend allowing  
5 portion 31 for allowing the fastening bolt 16 to move towards the front of the vehicle via the open side end 13a (the gap  $g$ ) when the secondary collision occurs. Namely, the bend allowing portion 31 is constructed to make its flexural deformation as shown  
10 in FIG. 21B when a predetermined load is applied thereto upon the secondary collision.

Note that the restricting member may extend towards the rear side of the vehicle from the front-  
of-the-vehicle side of the tilt adjusting groove 13.

15 Accordingly, at the early stage of the secondary collision, as shown in FIG. 21B, in the impact absorbing type steering column apparatus for the automotive vehicle according to the seventh embodiment, the vertical wall portion 12 and the  
20 column fastening/fixing portion 14 of the car body sided upper bracket 4 get collapsed while making their flexural deformations so as to rotate around the bending portion 11 (fulcrum), thereby absorbing the secondary impact energy.

25 Next, also at the later stage of the secondary collision, the car body sided upper bracket 4 consecutively gets collapsed while making its

flexural deformation so as to rotate around the bending portion 11 (fulcrum), thereby absorbing the secondary impact energy.

Simultaneously, the steering column 1 is so constructed as to be further movable beyond the tilt adjusting range as follows.

To be specific, at this later stage of the secondary collision, the bend allowing portion 31 of the restricting member 30 is bent or broken by the fastening bolt 16, thus allowing the fastening bolt 16 to move towards the front of the vehicle.

With this contrivance, at the later stage of the secondary collision, the fastening bolt 16 comes off the open end 13a (the gap g) of the tilt adjusting groove 13, thereby enabling the steering column 1 to be released from the car body. Accordingly, before the bending portion 11 of the car body sided upper bracket 4 reaches a limit of the flexure, the steering column 1 can be released from the car body, and a sufficient collapse stroke can be ensured.

(Eighth Embodiment)

FIG. 22A is an enlarged side view of the car body sided upper bracket (tilt bracket) of the impact absorbing type steering column apparatus according to an eighth embodiment of the present invention. FIG. 22B is a view showing a behavior of the car body sided upper bracket illustrated in FIG. 22A when

secondary collision happens.

In the eighth embodiment, the car sided upper bracket 4 is formed with the restricting member 30 extending substantially in the vertical direction of the vehicle so as to form a gap g between an obliquely-formed open side end 13a of the tilt adjusting groove 13 and the restricting member 30 itself. This restricting member 30 cooperates with the obliquely-formed open end 13a to delimit substantially the end of the tilt adjusting range of the tilt adjusting groove 13.

Further, the gap g is set smaller than a diameter of the fastening bolt 16. Moreover, the restricting member 30 is formed with the bend allowing portion 31 for allowing the fastening bolt 16 to move towards the front of the vehicle via the open side end 13a (the gap g) when the secondary collision occurs. Namely, on both sides of this bend allowing portion 31, a circular-arc-recessed portion 31a is formed outwardly, while a groove-like recessed portion 31b is formed inwardly. A contrivance is that these recessed portions 31a, 31b help the bend allowing portion 31 make its flexural deformation as shown in FIG. 22B when a predetermined load is applied thereto upon the secondary collision.

Note that only any one of the recessed portions 31a, 31b may be formed.

Hence, at the early stage of the secondary collision, as shown in FIG. 22B, in the impact absorbing type steering column apparatus for the automotive vehicle according to the eighth embodiment, the vertical wall portion 12 and the column fastening/fixing portion 14 of the car body sided upper bracket 4 get collapsed while making their flexural deformations so as to rotate around the bending portion 11 (fulcrum), thereby absorbing the secondary impact energy.

10           Next, also at the later stage of the secondary collision, the car body sided upper bracket 4 consecutively gets collapsed while making its flexural deformation so as to rotate around the bending portion 11 (fulcrum), thereby absorbing the secondary impact energy.

15           Simultaneously, the steering column 1 is so constructed as to be further movable beyond the tilt adjusting range as follows.

20           To be specific, at this later stage of the secondary collision, the bend allowing portion 31 of the restricting member 30 is bent or broken by the fastening bolt 16, thus allowing the fastening bolt 16 to move towards the front of the vehicle.

25           With this contrivance, at the later stage of the secondary collision, the fastening bolt 16 comes off the open end 13a (the gap g) of the tilt adjusting groove 13, thereby enabling the steering column 1 to



be released from the car body. Accordingly, before the bending portion 11 of the car body sided upper bracket 4 reaches a limit of the flexure, the steering column 1 can be released from the car body, and a sufficient collapse stroke can be ensured.

(Ninth Embodiment)

FIG. 23 is a side view of the impact absorbing type steering column apparatus for the automotive vehicle according to a ninth embodiment of the present invention.

FIG. 24 is a sectional view taken along the line XII-XII in FIG. 23.

Provided in the ninth embodiment is a column support member 40 attached to the car body sided upper bracket 4 and extending so as to be bent substantially in a U-shape under the steering column 1.

The column support member 40 is constructed to roughly delimit substantially the lower end of the tilt adjusting range and to prevent the steering column 1 from coming off downwards upon the secondary collision.

Namely, the ninth embodiment does not adopt the protrusion (stopper) S in the tilt adjusting groove 13. The column support member 40 described above is provided as a substitute for this protrusion S.

The column support member 40 is a U-shaped wire

and includes a pair of hook portions 41 of which front side ends are bent. The pair of hook portions 41 are secured into (engaged with) two pieces of securing holes 42 formed in the vertical wall portion 12 of the car body sided upper bracket 4.

A U-shaped surface of the column support member, which faces the undersurface of the steering column 1, works to determine a lower limit position of the tilt adjustment when moving the steering column 1 downwards for the tilt operation.

From what has been discussed so far, at the early stage of the secondary collision, in the impact absorbing type steering column apparatus for the automotive vehicle according to the ninth embodiment, the vertical wall portions 12 and the column fastening/fixing portions 14 of the car body sided upper bracket 4 get collapsed while making their flexural deformations so as to rotate around the bending portions 11 (fulcrum), thereby absorbing the secondary impact energy.

Next, at the later stage of the secondary collision, the open ends 13a of the tilt adjusting grooves 13 are directed to the front of the vehicle, and hence the fastening bolt 16 further moves beyond the tilt adjusting range and comes off the open ends 13a, thereby enabling the steering column 1 to be released from the car body.

Simultaneously, the column support wire member 40 prevents the steering column 1 from falling down.

To be specific, the fastening bolt 16 within the tilt adjusting groove 13 moves, and the lower portion of the steering column 1 abuts on the U-shaped surface of the column support member 40. The undersurface of the steering column 1 is sustained on the U-shaped surface of the column support wire member 40, in which state the steering column 1 continues to move forwards. Namely, the steering column 1 continues to move forwards a long distance while ensuring a collapsing trajectory without falling down. As a result, the collapse stroke can be further extended.

Accordingly, before the bending portion 11 of the car body sided upper bracket 4 reaches a limit of the flexure, the steering column 1 can be released from the car body, and a sufficient collapse stroke can be ensured. Moreover, the steering column 1 can move a long distance without falling down, and therefore the collapse stroke can be further extended.

The various modifications may be made in a variety of forms without being limited to the fifth through ninth embodiments discussed above.

For instance, a construction may be adopted, wherein the cut-away portion of the car body sided lower bracket (or the column sided lower bracket) is

provided with a protrusion, thereby absorbing the impact load.

Further, for example, in the fifth through ninth embodiments discussed above, without being limited to  
5 the upper and lower side brackets, the vertical wall portion may be disposed in front or in rear of the car body fitting portion, and the column fastening/fixing portion may be disposed in front or in rear of the vertical wall portion. Moreover, the  
10 hinge fixing portion may be disposed in front or in rear of the vertical wall portion.

Still further, the embodiments discussed above may be carried out singly or in combination.

Note that the discussion on each of the  
15 embodiments has been focused on the steering column apparatus incorporating the tilt mechanism, however, the present invention can be applied to steering column apparatuses having none of the tilt mechanisms. Moreover, the steering column apparatus of which the  
20 interior has no means of absorbing the energy has been described, however, according to the present invention, the energy absorbing means may be provided within the steering column apparatus.

Furthermore, according to other embodiments, as  
25 in the fifth embodiment, the grease containing the extreme-pressure additive such as molybdenum disulfide, etc. may be applied to between the tilt

groove 13, the column fastening/fixing portion 14 of  
the car body sided upper bracket 4 and the nut 39 or  
the second cam 19, and between the bolt  
fastening/fixing portion 14 and the distance bracket  
15.

5

WHAT IS CLAIMED IS:

1. In an impact absorbing type steering column apparatus for an automotive vehicle, capable of adjusting a steering position and, when a secondary  
5 collision happens, absorbing impact energy thereof by moving a steering column supported through a bracket on a car body towards the front of the vehicle,

an improvement characterized in that said bracket includes a restricting portion for  
10 restricting a steering position adjusting range of said steering column, and

said restricting portion allows, upon the secondary collision, said steering column to move beyond the steering position adjusting range.

15

2. In an impact absorbing type steering column apparatus for an automotive vehicle, capable of adjusting a steering position and, when a secondary collision happens, absorbing impact energy thereof by  
20 moving a steering column supported through a bracket on a car body towards the front of the vehicle,

an improvement characterized in that said bracket includes a steering column position adjusting groove, through which a fastening member of said  
25 steering column is inserted and of which one end is opened, and a restricting portion for restricting a steering position adjusting range of said steering

column, and

said restricting portion allows, upon the secondary collision, said steering column to move beyond the steering position adjusting range.

5

3. An impact absorbing type steering column apparatus for an automotive vehicle according to claim 2, wherein said groove serves for adjusting a tilt position of said steering column, and a lower  
10 bracket supporting said steering column through a hinge mechanism in the front of the vehicle and supported on the car body, is provided on a front-of-the vehicle side of said bracket,

said lower bracket includes a cut-away portion  
15 through which a pivot of said hinge mechanism is inserted and of which a front-of-the-vehicle side is opened, and

said pivot comes off said open end of said cut-away portion upon an axis-directional input of said  
20 steering column when the secondary collision happens, and said steering column is released from said lower bracket.

4. An impact absorbing type steering column  
25 apparatus for an automotive vehicle according to claim 2 or 3, wherein a protrusion for regulating a movement of said fastening member is provided as said

restricting portion within said adjusting groove.

5. An impact absorbing type steering column apparatus for an automotive vehicle according to  
5 claim 4, wherein said protrusion is constructed of a plurality of protrusions formed in alignment in their directions towards the front of the vehicle.

6. An impact absorbing type steering column  
10 apparatus for an automotive vehicle according to claim 4 or 5, wherein said protrusion includes an abutting surface on the side facing said fastening member.

15 7. An impact absorbing type steering column apparatus for an automotive vehicle according to claim 3, wherein said restricting portion of said bracket extends substantially in front-and-rear directions of the vehicle in a way that leaves said  
20 open end, and is formed to delimit substantially a lower portion of said position adjusting groove, and said restricting member includes a bend allowing portion for allowing said fastening member of said steering column to move towards the front of the  
25 vehicle through said open end.

8. An impact absorbing type steering column



apparatus for an automotive vehicle according to  
claim 3, wherein said restricting portion of said car  
body sided bracket extends substantially in vertical  
directions in a way that leaves said open end, and is  
5 formed to delimit substantially a side position of  
said adjusting groove, and

said restricting portion includes a bend  
allowing portion for allowing said fastening member  
of said steering column to move towards the front of  
10 the vehicle through said open end.

9. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
claim 2 or 3, further comprising a column support  
15 member extending so as to be curved under said  
steering column,

wherein said column support member delimits  
substantially the lower portion of the steering  
position adjusting range, and prevents said steering  
20 column from falling down.

10. In an impact absorbing type steering column  
apparatus for an automotive vehicle, capable of  
adjusting a steering position and, when a secondary  
25 collision happens, absorbing impact energy thereof by  
moving a steering column supported through a bracket  
on a car body towards the front of the vehicle,

an improvement characterized in that there is provided a restricting member including a first restricting portion and a second restricting portion, said restricting member allows, within said first  
5 restricting portion, said steering column to move for a positional adjustment, then deforms when said steering column moves, upon a secondary collision, beyond a first predetermined range restricted by said first restricting portion, and restricts the movement  
10 of said steering column within a second predetermined range by use of said second restricting portion.

11. An impact absorbing type steering column apparatus for an automotive vehicle according to  
15 claim 10, wherein said bracket is constructed of an upper bracket and a lower bracket, a bolt is inserted through a hole of said upper bracket, and said steering column is supported by said upper bracket,

said restricting member is formed integrally  
20 with said car body sided upper bracket,

said first restricting portion is formed with said hole, and

when said steering column moves through only the first predetermined range upon the secondary  
25 collision, said bolt causes said restricting member to deform and enters said second restricting portion provided adjacent to said first restricting portion.

12. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
claim 11, wherein when said bolt enters said second  
5 restricting portion, said restricting member makes  
its flexural deformation so as to extend in a moving  
direction of said bolt.

13. An impact absorbing type steering column  
10 apparatus for an automotive vehicle according to  
claim 11, wherein said second restricting portion is  
previously formed as an elongate hole suitable for  
guiding said bolt in its moving direction when said  
bolt has entered said second restricting portion.

15

14. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
claim 11, wherein said hole of said upper bracket is  
a groove for a tilt adjustment, said bolt is a  
20 fastening bolt for the tilt adjustment, and said  
lower bracket pivotally supports said steering column.

15. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
25 claim 11, wherein a bolt is inserted through a hole  
of said lower bracket, and said steering column is  
supported by said lower bracket,

said restricting member is formed integrally  
with said car body sided lower bracket,

said first restricting portion is formed with  
said hole, and

5           when the secondary collision happens, impact  
energy is absorbed in a way that causes a flexural  
deformation of said restricting member while moving  
said steering column towards the front of the vehicle,  
and

10           when said steering column moves through only the  
first predetermined range, said bolt causes said  
restricting member to deform and enters said second  
restricting portion provided adjacent to said first  
restricting portion.

15

16. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
claim 15, wherein when said bolt enters said second  
restricting portion, said restricting member makes  
20   its flexural deformation so as to extend in a moving  
direction of said bolt.

17. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
25   claim 15, wherein said second restricting portion is  
previously formed as an elongate hole suitable for  
guiding said bolt in its moving direction when said

bolt has entered said second restricting portion.

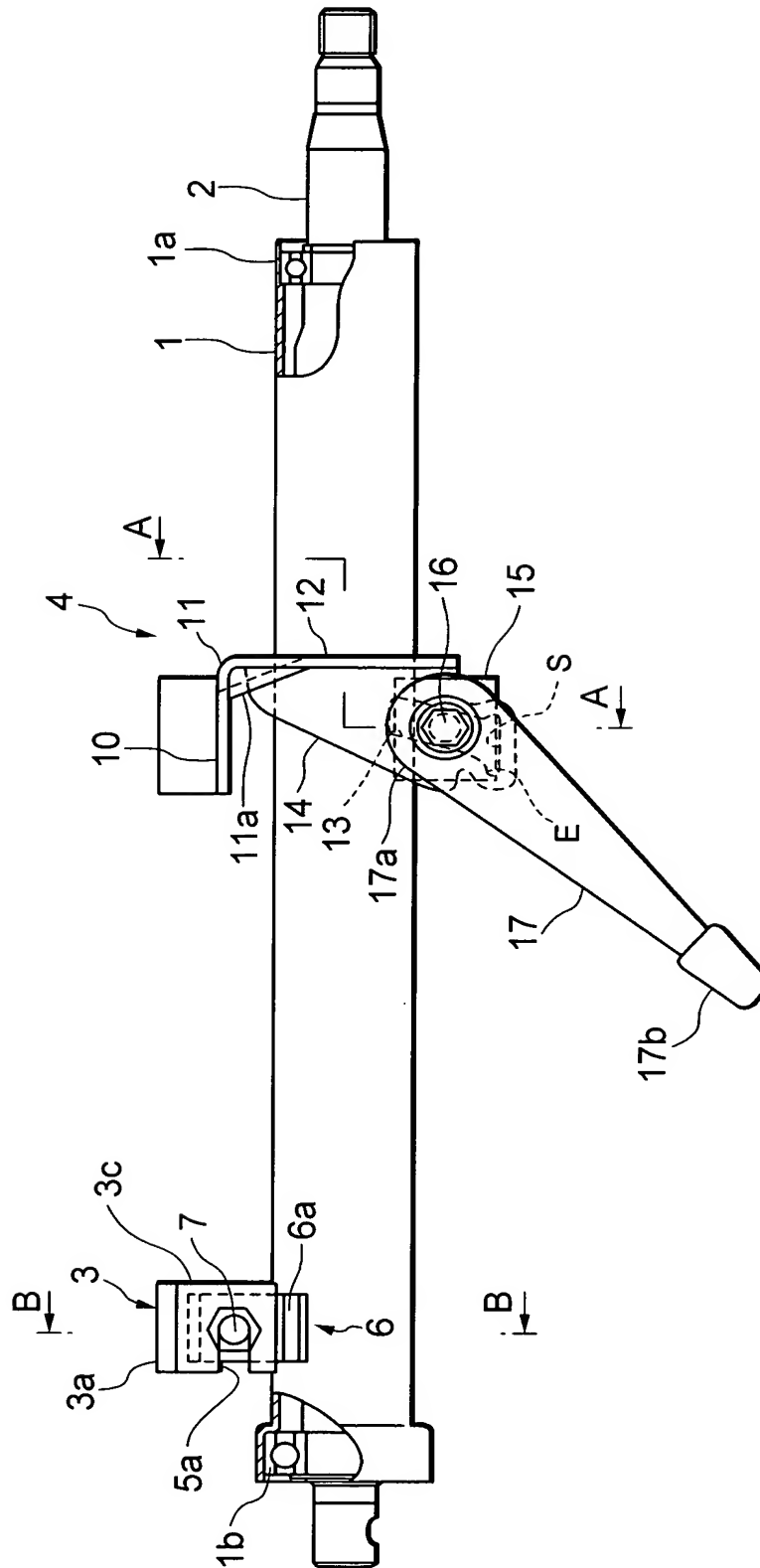
18. An impact absorbing type steering column  
apparatus for an automotive vehicle according to  
5 claim 15, wherein said hole of said car body sided  
lower bracket is a support hole for the tilt  
adjustment, and said bolt is a tilt adjusting hinge  
pin for determining a tilt center when inserted into  
said support hole.

Abstract

An impact absorbing type steering column apparatus for an automotive vehicle is capable of adjusting a steering position and, when a secondary  
5 collision happens, absorbs impact energy thereof by moving a steering column supported through a bracket on a car body towards the front of the vehicle. The bracket includes a restricting portion for restricting a steering position adjusting range of  
10 the steering column. The restricting portion allows, upon the secondary collision, the steering column to move beyond the steering position adjusting range.

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FIG. 1



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FIG. 2

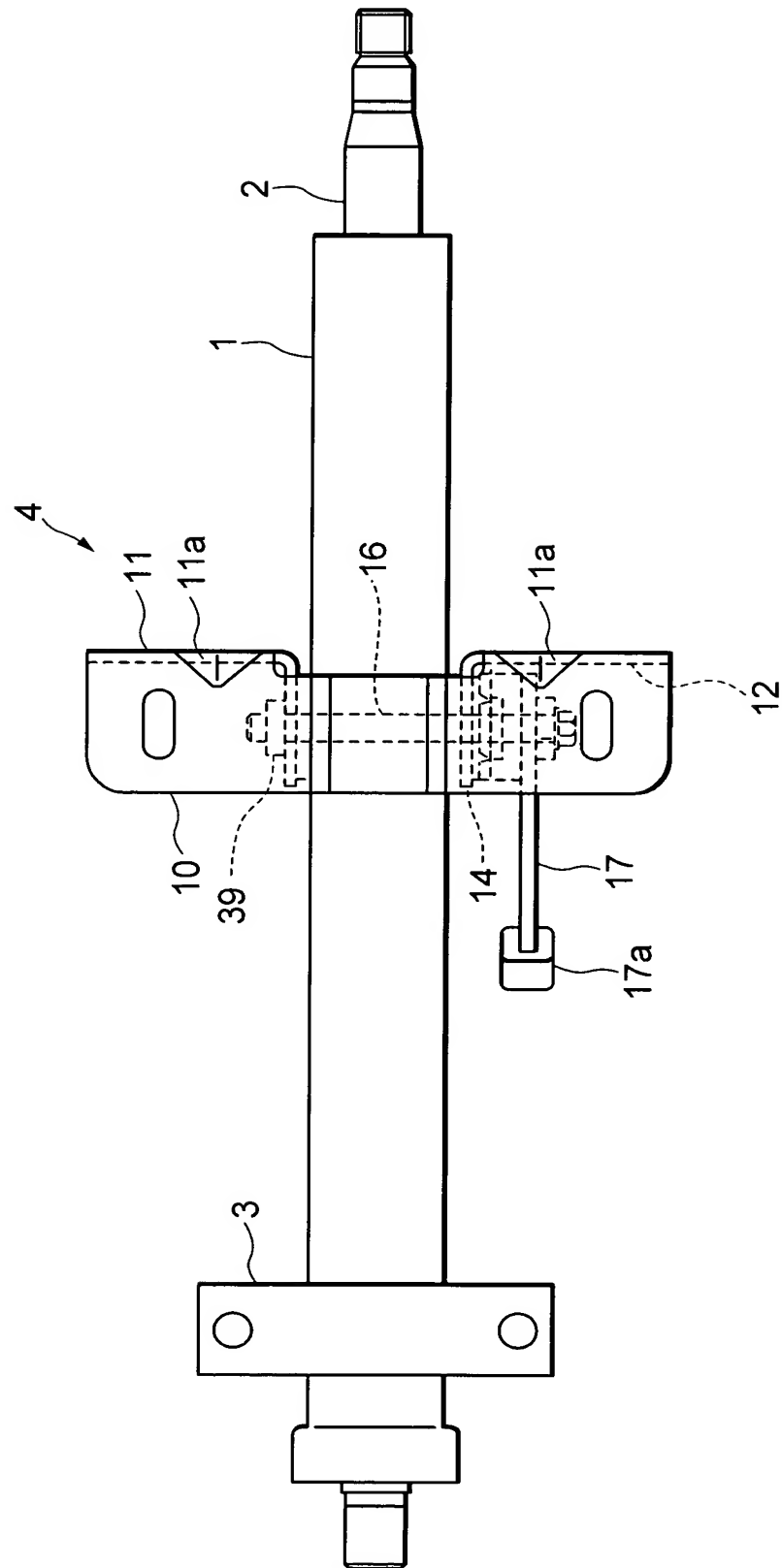
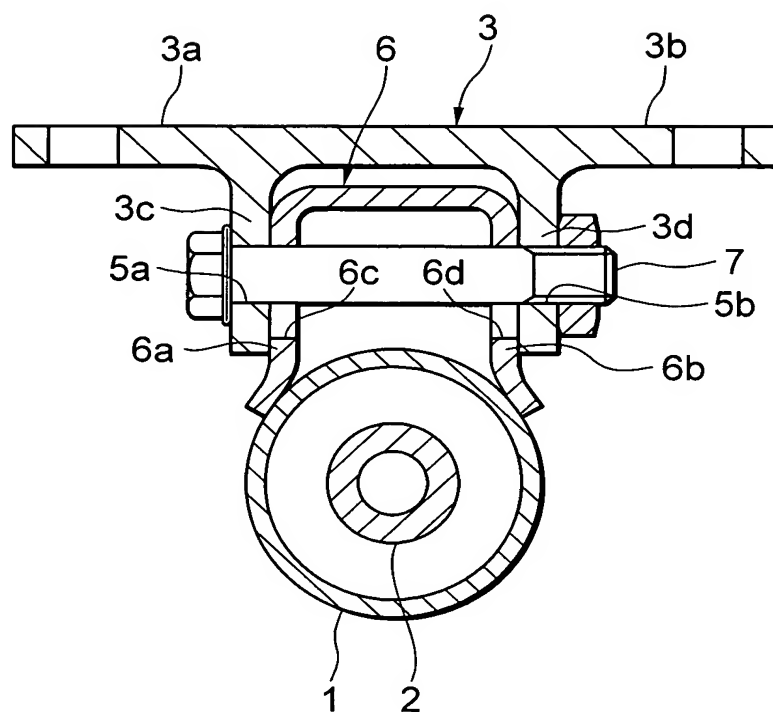






FIG. 4



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FIG. 5

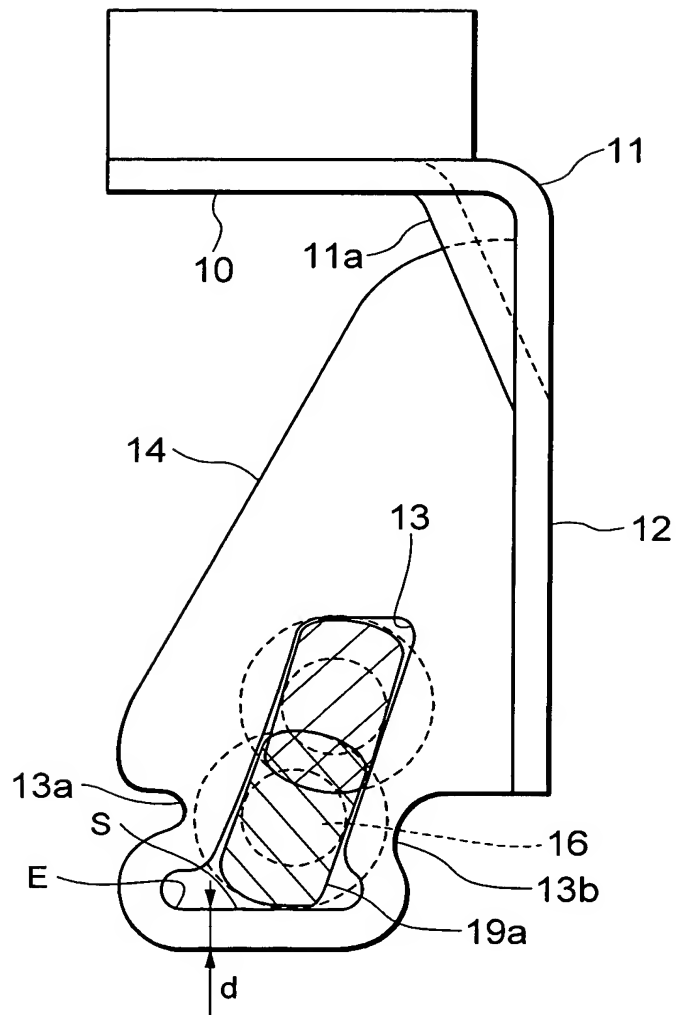


FIG. 6

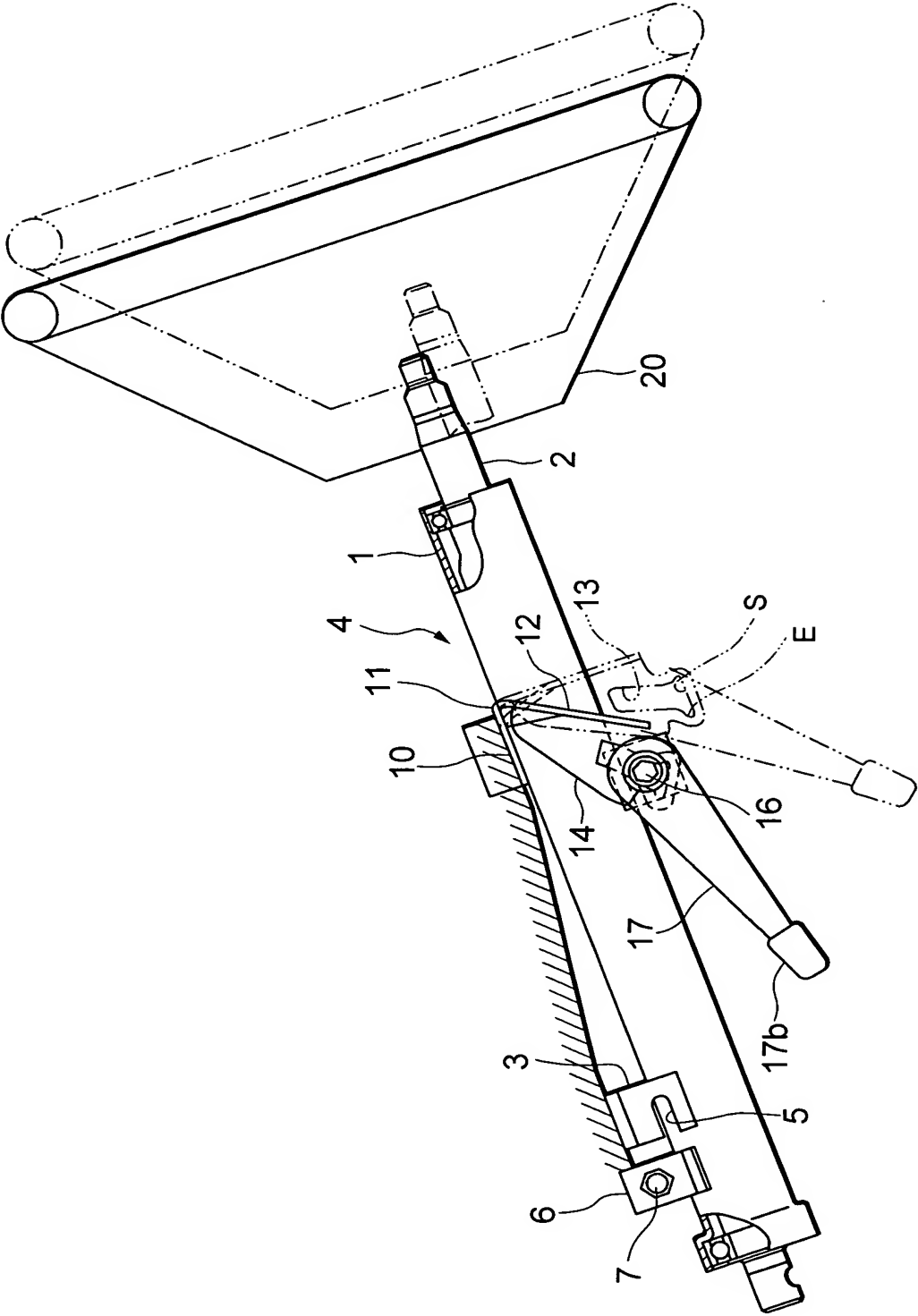
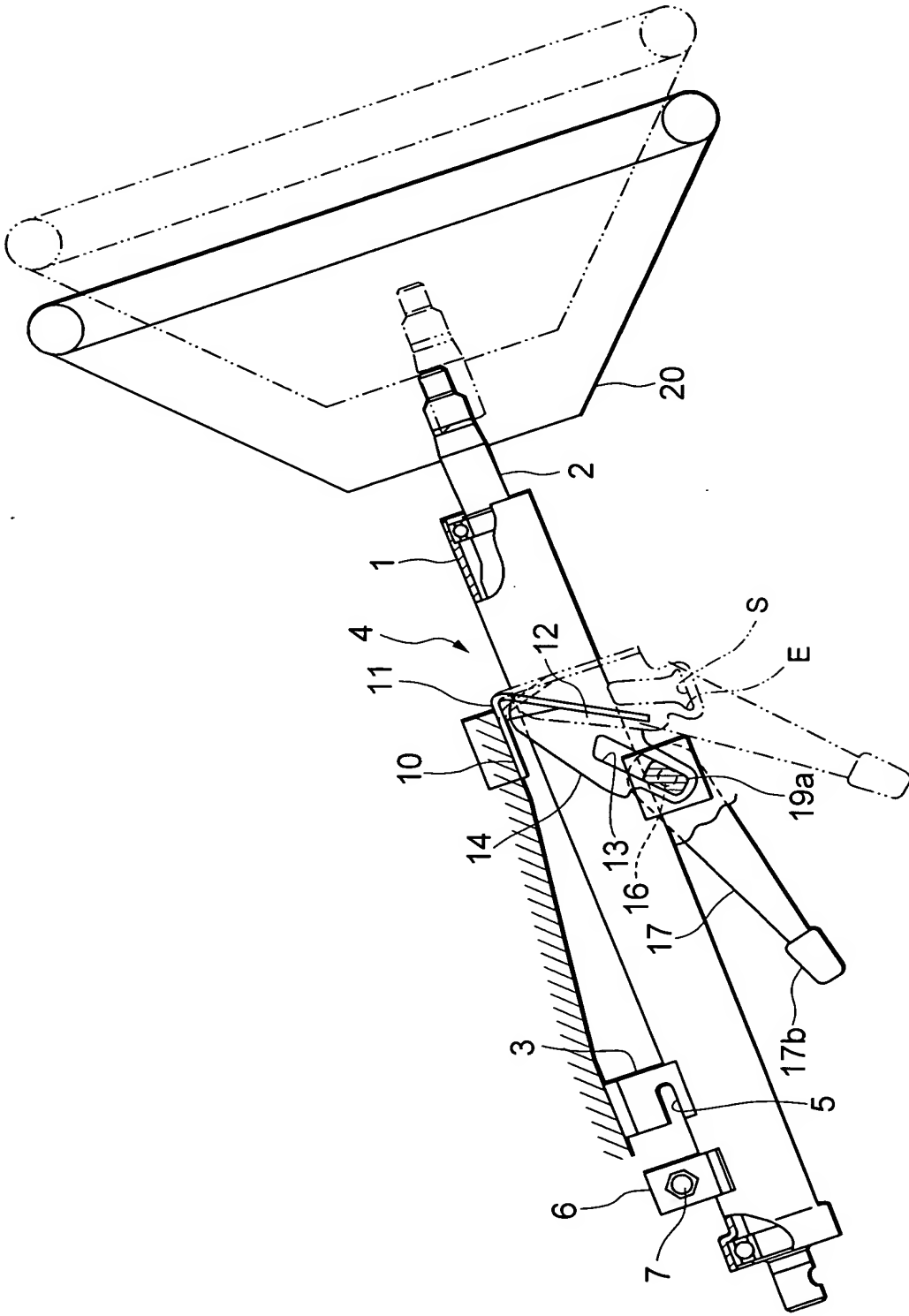


FIG. 7



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FIG. 8

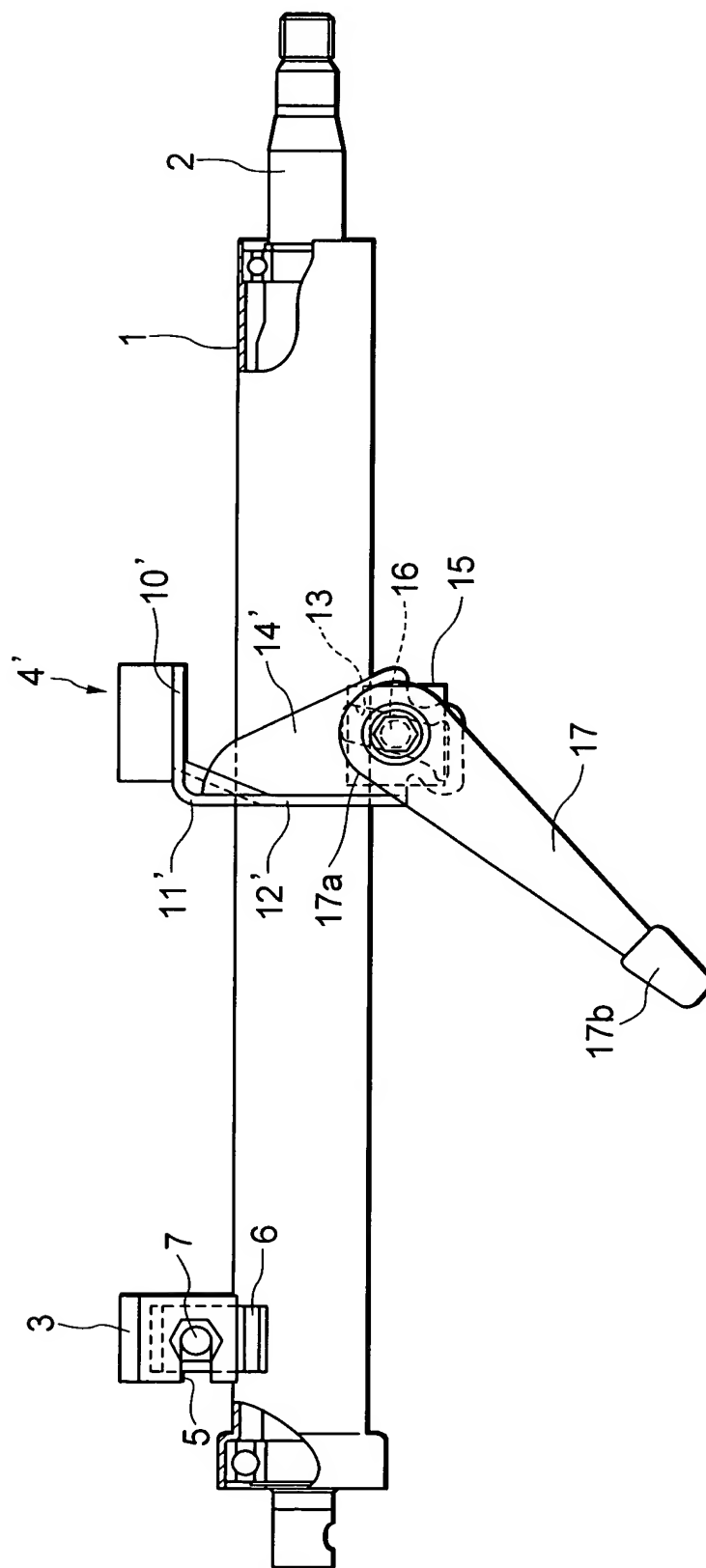


FIG. 9

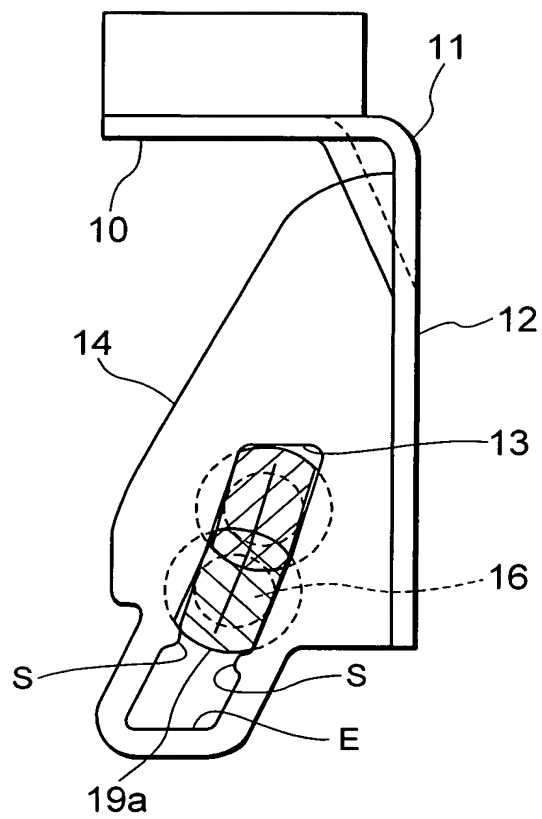


FIG. 10

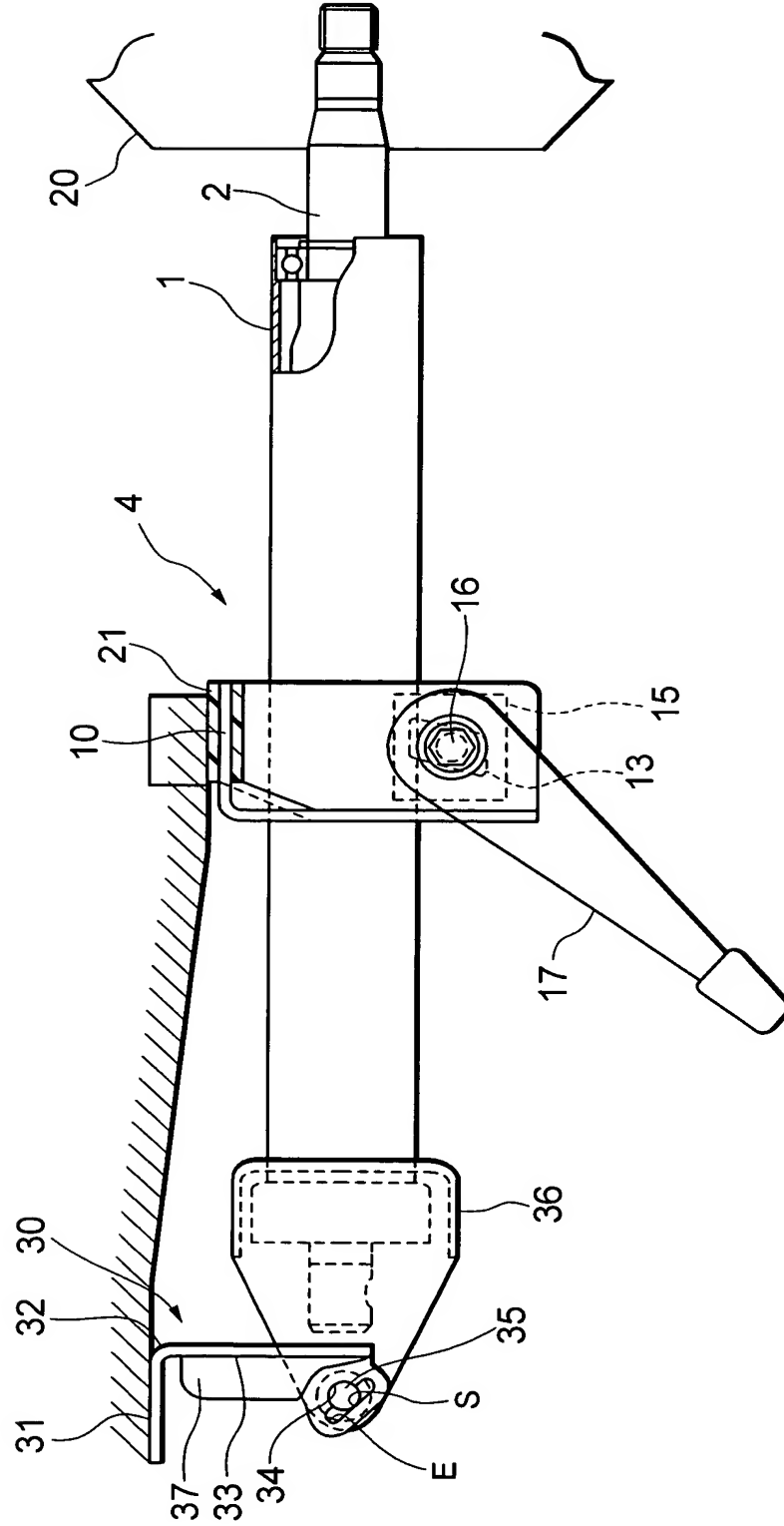






FIG. 12

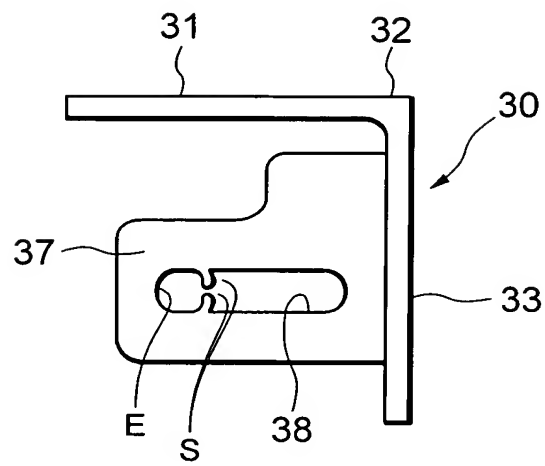
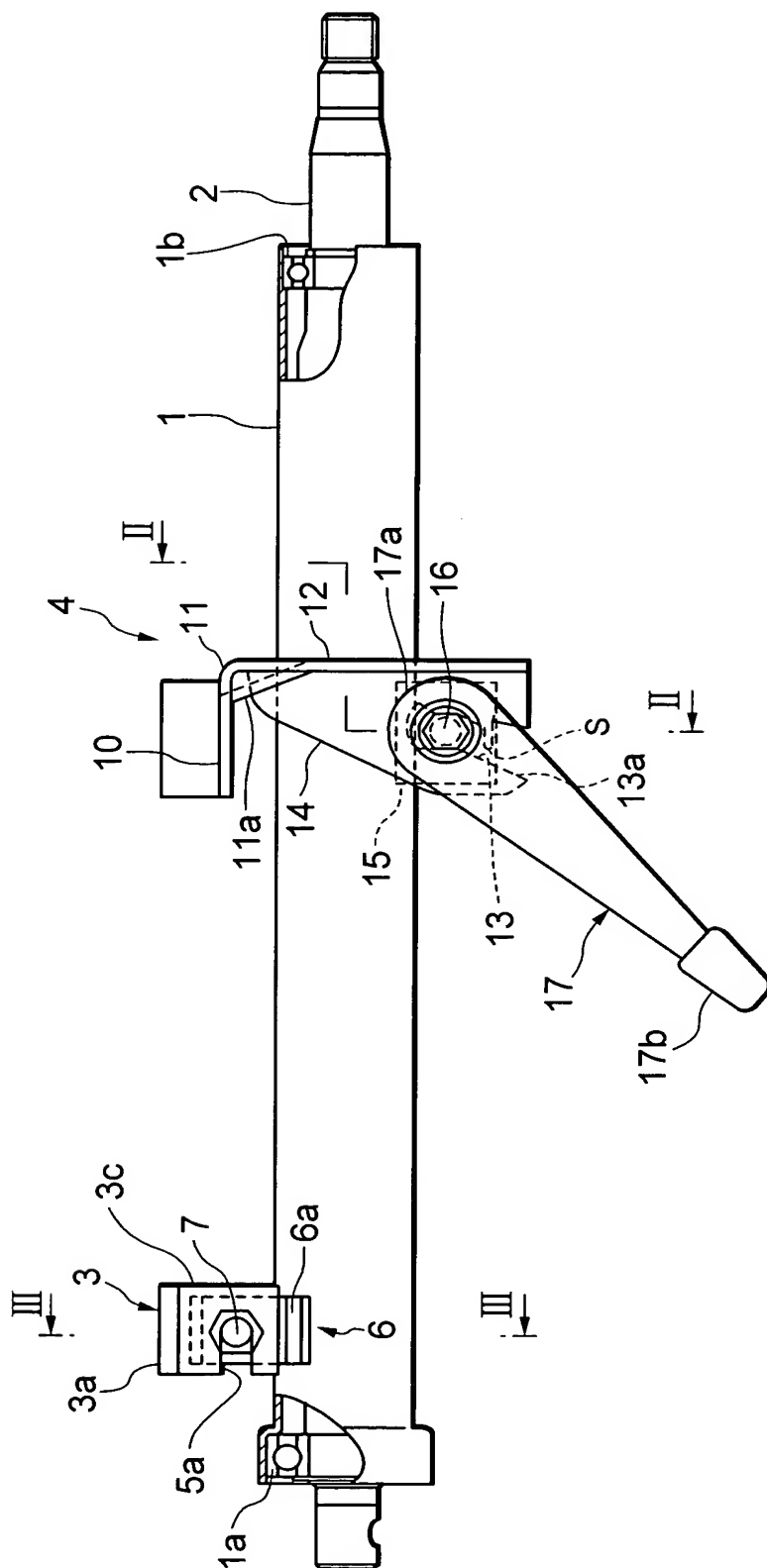


FIG. 13



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FIG. 14A

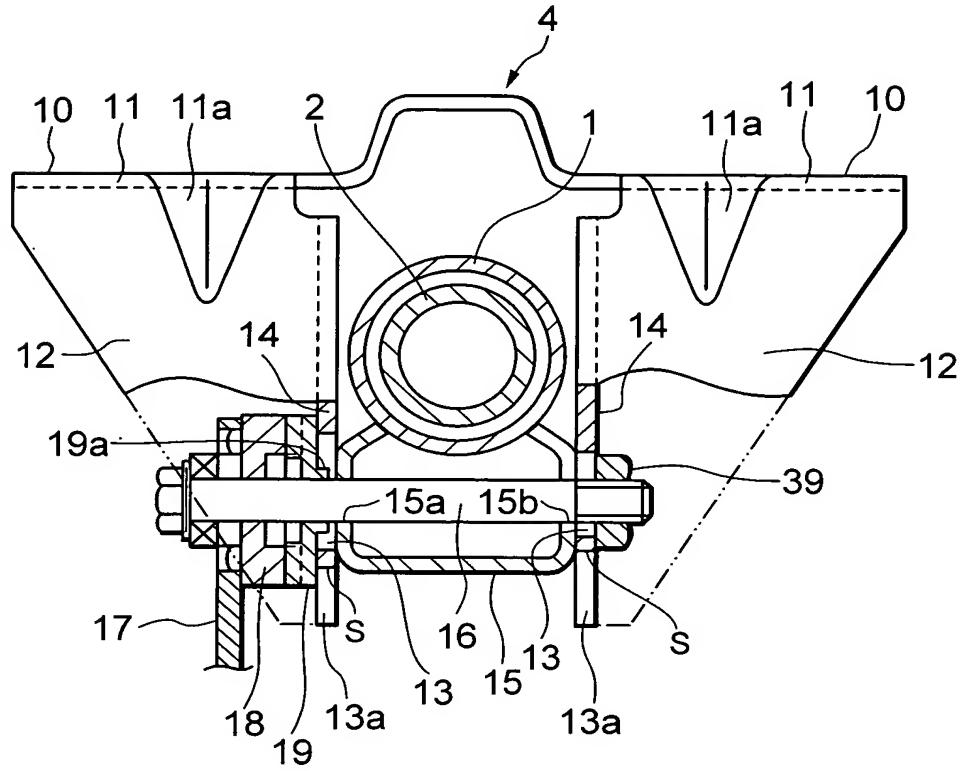


FIG. 14B

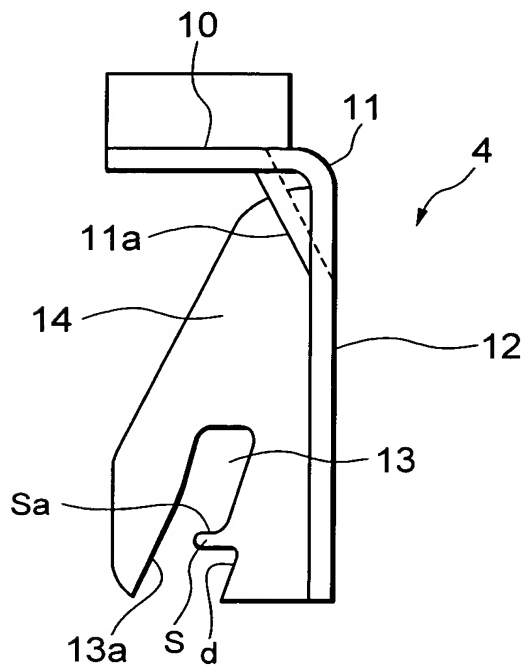


FIG. 15

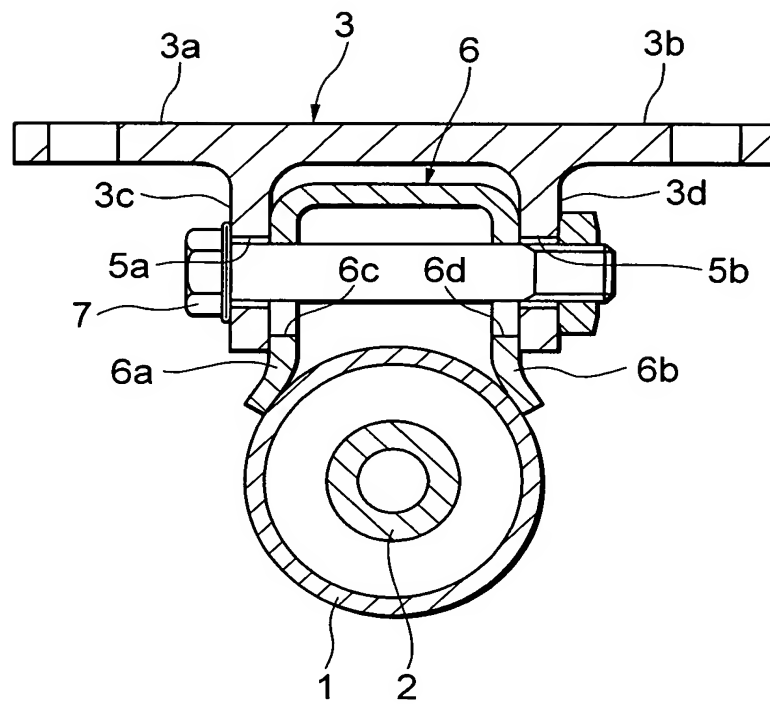


FIG. 16

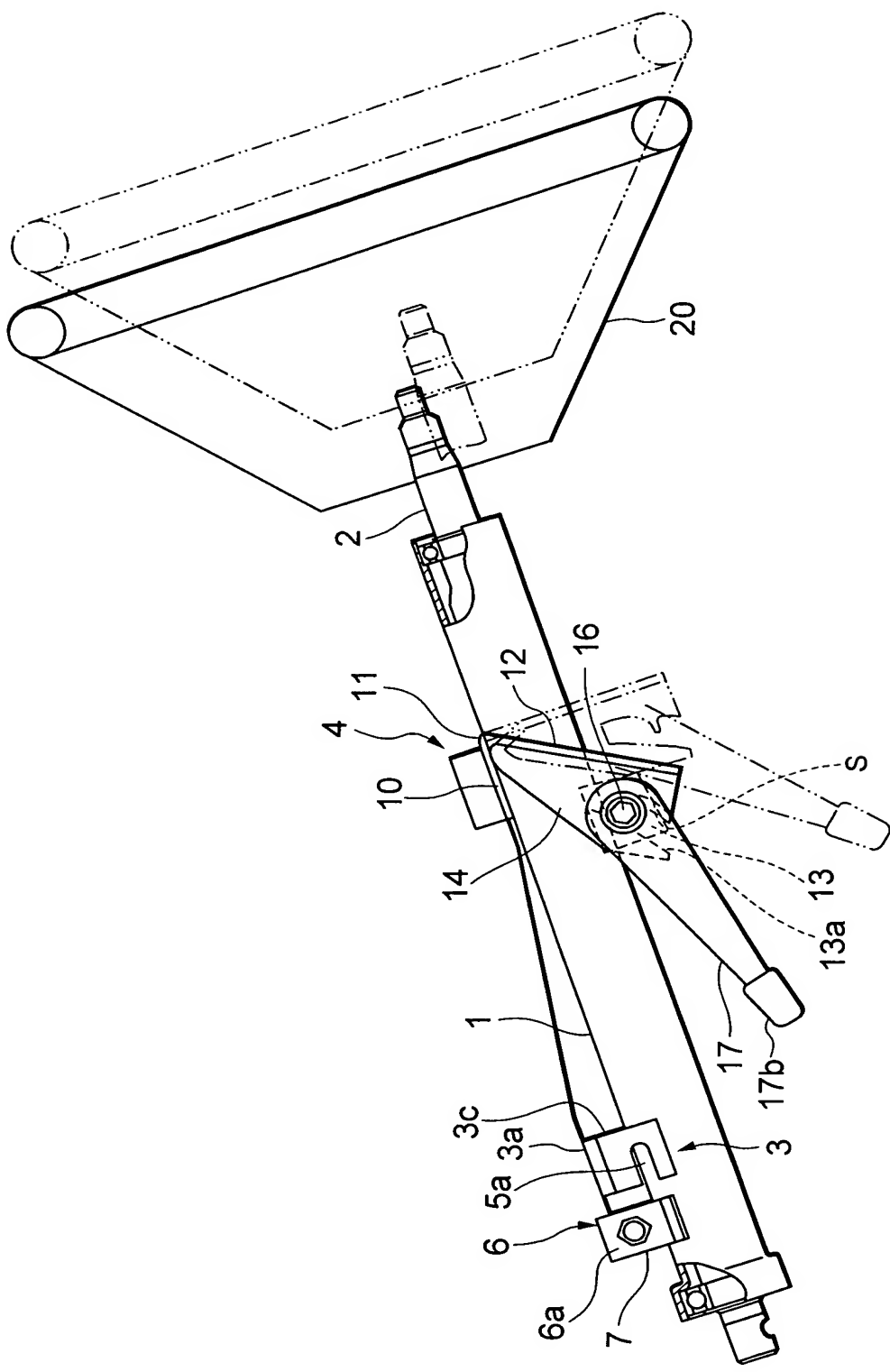




FIG. 18

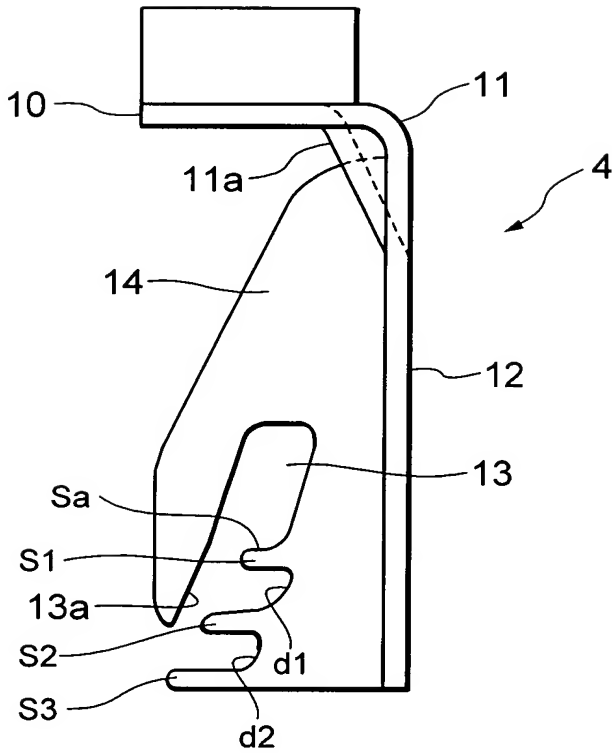




FIG. 19

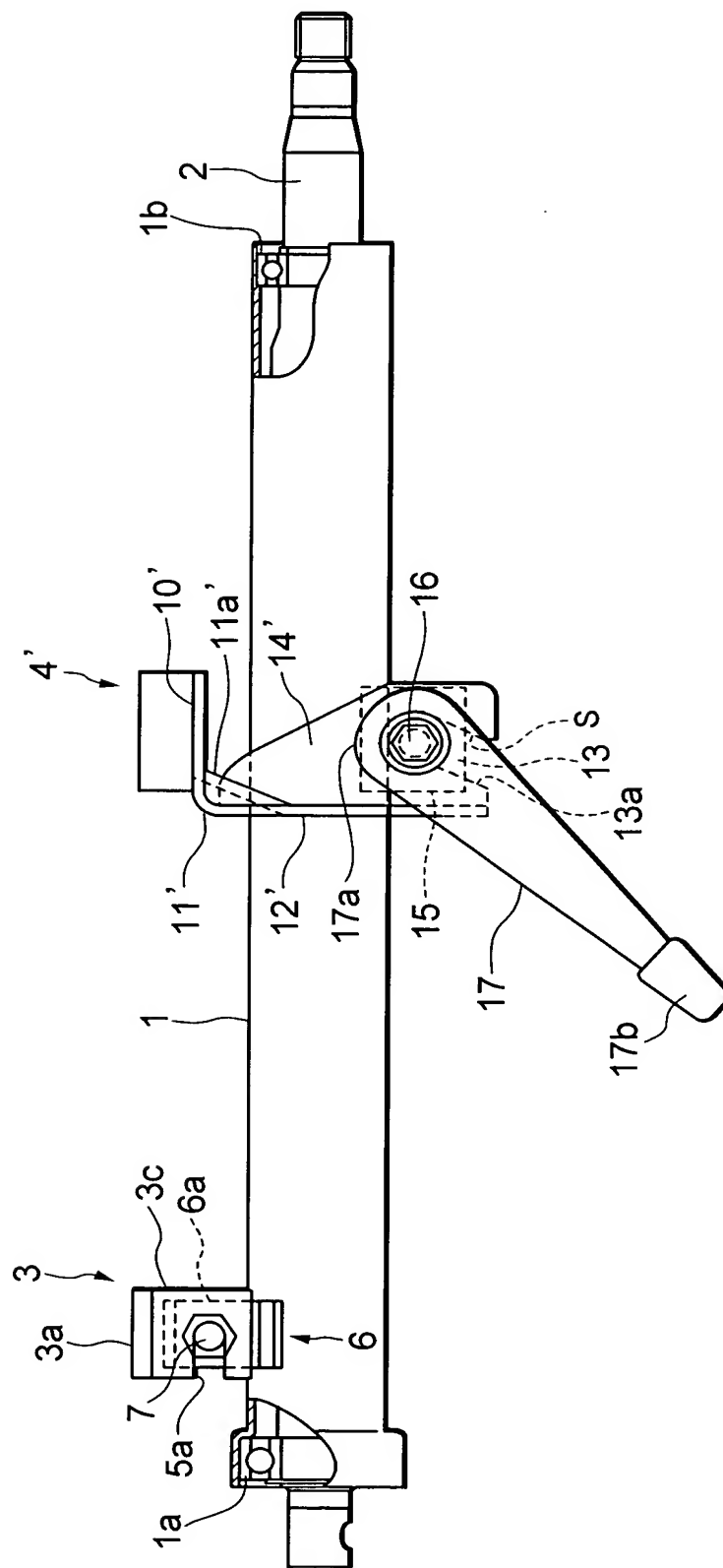


FIG. 20

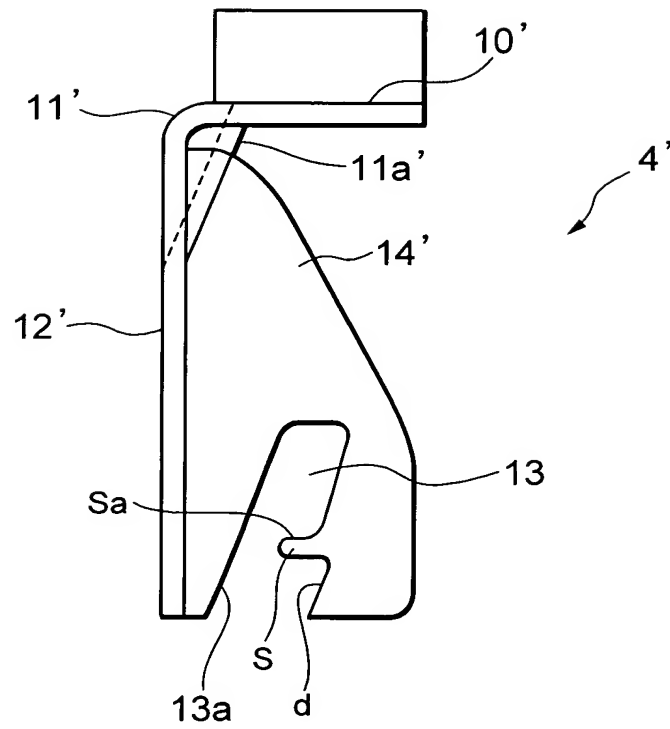


FIG. 21A

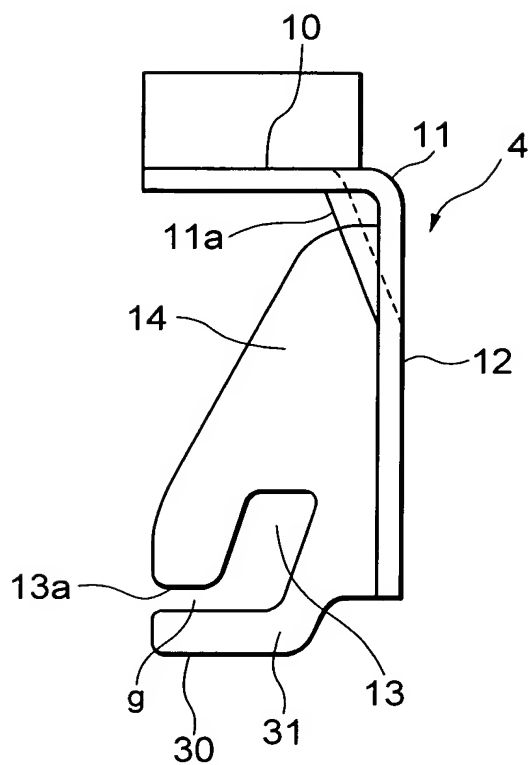


FIG. 21B

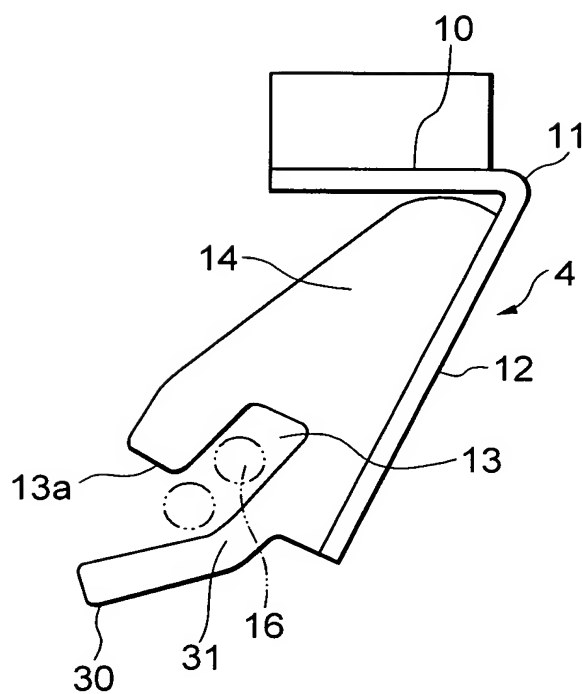


FIG. 22A

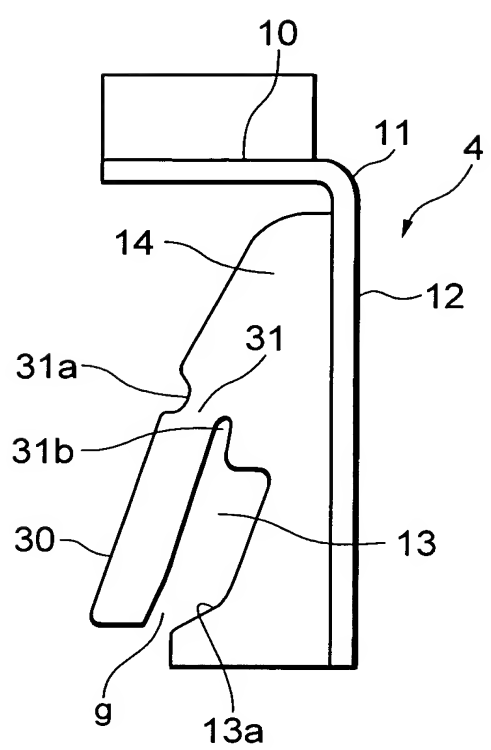


FIG. 22B

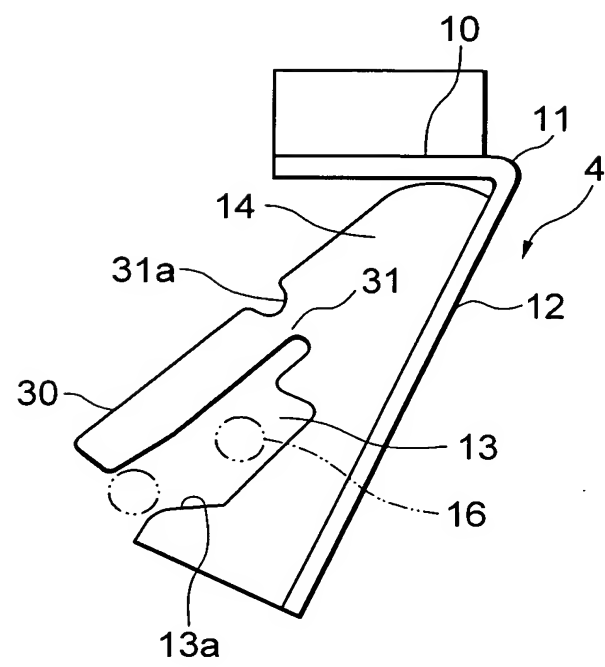




FIG. 24

